

## Matlab Code

- This code has two parts: (i) beam and (ii) plate.
- To run a beam problem copy the BEAM.m function and run in Matlab. The required functions to run BEAM.m are: aplybcdyn.m, FEAPLYC2.m, FEAPLYCS.m, FEASMBL1.m, FEASMBL2.m, feasmbf.m, FEBEM.m, FELDOF.m, FEELDOF.m, feload.m, felresp.m, yprime\_beam.m, optiactuator.m.
- To run a plate problem copy the Plate\_D\_F.m function and run in Matlab. The required functions are: aplybcdyn.m, BOUNRY.m, ELKMFR\_D\_F.m, ELSTF.m, FEAPLYC2.m, FEAPLYCS.m, FEASMBL1.m, FEASMBL2.m, FELDOF.m, femesh.m, MSH2DR.m, Piezo\_resultant.m, PLSTF.m, QBARSTF.m, QBARPIEZO.m, SHPRCT.m, STRESS.m, yprime\_plate.m.

### PART1 PLATE

Run plate\_D\_F.m, 2. Give an input file (example circle.inp) give an output file (example circle.out).

1. All the displacements and stress values will be stored in filename.out
2. Plots will be displayed
3. Read through the programs to change the working sequence
4. Read through the programs and input template to use the automated mesh code

### Plate\_D\_F.m

```
%-----  
% PLATE FINITE ELEMENT PROGRAM  
% Dynamic Analysis of composite plates  
% CLPT, FSDT CAPABILITIES  
% RECTANGULAR ISO PARAMETRIC ELEMENTS  
%-----  
clear all  
close all  
FILE1 = input('Input Data File Name ','s');  
LINP = fopen(FILE1,'r');  
FILE2 = input('Output Data File Name ','s');  
stsr_time=cputime;  
LOUT = fopen(FILE2,'w');  
global LINP LOUT % Files  
global E1 E2 G12 G13 G23 ANU12 RHO ALFA1 ALFA2 THETA THKNS MATYP %  
Material  
global ELF ELK ELM ELXY ELU ELV ELA A1 A2 A3 A4 A5 Q0 QX QY % Stiff1  
global A B D E F G H A44 A45 A55 D44 D45 D55 % Stiff2  
global Q QBAR ALFA Z % Elastic  
global IPDF IPDR NIPF NIPR % Point  
global P0 P1 P2 P3 P4 P5 P6 % Integ  
global T0 T1 TN TM TL TK % Therm  
global SF GDSF SFH GDSFH GDSFH % Shape  
global PIEZO_E PM PN % Piezo  
  
%-----  
% DATA INPUT  
%-----
```

```

INITIAL=0;
TITLE = fgets(LINP); %READ TITLE
TMP = str2num(fgets(LINP)); %Read THEORY and ANALYSIS type
[ITYPE, IDYN, N THERM, NEIGN, PIEZO] =
deal(TMP(1), TMP(2), TMP(3), TMP(4), TMP(5));
if IDYN==0 NEIGN=0; end
TMP = str2num(fgets(LINP)); %READ LAYERS, NTHKNS, NUMAT
[LAYERS, NTHKNS, NUMAT] = deal(TMP(1), TMP(2), TMP(3));
TMP = str2num(fgets(LINP)); %READ THETA for all LAYERS
for I=1:LAYERS THETA(I)=TMP(I); end
TMP = str2num(fgets(LINP)); %READ THKNS for all LAYERS
for I=1:LAYERS THKNS(I)=TMP(I); end
TMP = str2num(fgets(LINP)); %READ MATerialTYPe for all LAYERS
for I=1:LAYERS MATYP(I)=TMP(I); end

if PIEZO>0
TMP = str2num(fgets(LINP)); %Read piezo values
[NPLY, p_tk, p_theta, AV] = deal(TMP(1), TMP(2), TMP(3), TMP(4));
end

%READ Material Properties for NUMAT
for N=1:NUMAT
TMP = str2num(fgets(LINP));
[E1(N), E2(N), G12(N), G13(N), G23(N), ANU12(N)] =
deal(TMP(1), TMP(2), TMP(3), TMP(4), TMP(5), TMP(6));
if IDYN>0 TMP = str2num(fgets(LINP)); RHO(N)=deal(TMP(1)); end
if N THERM>0
TMP=str2num(fgets(LINP));
[ALFA1(N), ALFA2(N)] = deal(TMP(1), TMP(2));
end
end
if PIEZO>0
TMP = str2num(fgets(LINP));
[EC13, EC23, EC33, EC41, EC42, EC51, EC52] = deal(TMP(1), TMP(2), TMP(3),
TMP(4), TMP(5), TMP(6), TMP(7));
end

%READ AK
if ITYPE>1 AK=fscanf(LINP, '%e\n', 1); else AK=0; end %else condition
added
TMP = str2num(fgets(LINP)); %READ XL, YL
[XL, YL] = deal(TMP(1), TMP(2));
TMP = str2num(fgets(LINP)); %READ DH for NTHKNS
for I=1:NTHKNS DH(I)=TMP(I); end
if NEIGN==0
TMP=str2num(fgets(LINP));
[LOAD, LPT]=deal(TMP(1), TMP(2));
TMP=str2num(fgets(LINP)); %READ Q0, QX, QY
[Q0, QX, QY]=deal(TMP(1), TMP(2), TMP(3));
end
if N THERM>0
TMP=str2num(fgets(LINP));
[T0, T1]=deal(TMP(1), TMP(2)); %READ T0, T1
end

% Read finite element mesh information:
TMP = fscanf(LINP, '%e\n', 5);

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[IELTYP,NPE,MESH,IGRAD,NPRNT] =
deal(TMP(1),TMP(2),TMP(3),TMP(4),TMP(5));
TMP = fscanf(LINP,'%e\n',3);
[IPDF,IPDR,ISTR] = deal(TMP(1),TMP(2),TMP(3));

if MESH==0
    TMP=fscanf(LINP,'%d',2);    %READ NEM,NNM
    [NEM,NNM]=deal(TMP(1),TMP(2));
    for N=1:NEM                %READ NOD for NPE
        TMP = fscanf(LINP,'%d',4);
        for I=1:NPE NOD(N,I)=TMP(I); end
    end
    for I=1:NNM
        TMP=fscanf(LINP,'%f',2);
        for J=1:2 GLXY(I,J)=TMP(J); end
    end
else
    TMP = fscanf(LINP,'%e\n',3);
    [NX,NY,EQSP] = deal(TMP(1),TMP(2),TMP(3));
    if EQSP==1 %EQSP=1 -- equaliy spaced mesh with NDI VX divisions
        TMP = fscanf(LINP,'%e\n',2);
        [X0,Y0]= deal(TMP(1),TMP(2));
        for I=1:NX DX(I)=XL/NX; end
        for I=1:NY DY(I)=YL/NY; end
    else %EQSP=0 -- unequaliy spaced mesh with DX()
        TMP = str2num(fgets(LINP));
        X0=TMP(1);
        for I=1:NX DX(I)=TMP(I+1); end
        TMP = str2num(fgets(LINP));
        Y0=TMP(1);
        for I=1:NY DY(I)=TMP(I+1); end
    end
end
if NPE<=4
    IEL=1;
else
    IEL=2;
end

if MESH>=1
    [NOD,GLXY,NNM,NEM]=MSH2DR(IEL,NX,NY,NPE,DX,DY,X0,Y0);
end

NDF=5;
if ITYPE==1 NDF=6; end
NEQ=NNM*NDF;
NN=NPE*NDF;
%bc=0 -> provide boundary conditions
%bc=1 -> auto genrates boundary conditions- provide edge conditions
%bc=2 -> special boundary conditions
bc=fscanf(LINP,'%d',1);
%Generating Boundary Conditions
%Edge conditions:: SS=1; FIXED=2
if bc==1
    TMP=fscanf(LINP,'%d',4);
    [bcx0,bcx1,bcy0,bcy1] = deal(TMP(1),TMP(2),TMP(3),TMP(4));

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```

[NSPV, ISPV, VSPV]=bcgen(NOD, GLXY, NNM, NEM, bcx0, bcxl, bcy0, bcyl, XL, YL);
% Eigen Boundary conditions
if NEIGN==0      NSSV=fscanf(LINP, '%e', 1);      end
if NSSV==0 ISSV=0; VSSV=0; end % this line is added to declare
ISSV, VSSV
elseif bc==0
% Read specified primary and secondary degrees of freedom: node
% number, local degree of freedom number, and specified value.
NSPV=fscanf(LINP, '%d', 1);
if NSPV~=0
for I=1:NSPV
TMP=fscanf(LINP, '%d', 2);
for J=1:2 ISPV(I, J)=TMP(J); end
end
if NEIGN==0
for I=1:NSPV
VSPV(I)=fscanf(LINP, '%e', 1);
end
end
end

if NEIGN==0      NSSV=fscanf(LINP, '%e', 1);      end
if NSSV~=0
for I=1:NSPV
TMP=fscanf(LINP, '%d', 2);
for J=1:2 ISPV(I, J)=TMP(J); end
end
for I=1:NSSV VSSV(I)=fscanf(LINP, '%e', 1); end
end
if NSSV==0 ISSV=0; VSSV=0; end % this line is added to declare
ISSV, VSSV
end %BC input selection ends
for I=1:NSPV
BCDOF(I)=ISPV(I, 1)*5+ISPV(I, 2);
end
if PIEZO>0
I=1:NEM; PIEZO_E_in(I)=0; %assumes no element has piezo
initially
% for I=1:NEM PIEZO_E_in(I)=fscanf(LINP, '%d', 1); end
% PIEZO_E(I)=fscanf(LINP, '%d', 1); %puts '1' for the elements
have piezo
% else
% % I=1:NEM; PIEZO_E(I)=0;
N_PIEZO_EL=fscanf(LINP, '%d', 1);
for I=1:N_PIEZO_EL PIEZO_EL(I)=fscanf(LINP, '%d', 1); end

for J=1:N_PIEZO_EL
for I=1:NEM
if I==PIEZO_EL(J);
PIEZO_E_in(I)=1;
end
end
end
end

% %
% % Dynamic Input commented

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% %           if IDYN~=0
% %             IDYN=2;
% %             if NEIGN==0
% %               TMP = str2num(fgets(LINP));
% %               [NTIME,NSTP,INTVL,INTIAL] =
deal(TMP(1),TMP(2),TMP(3),TMP(4));
% %               if INTVL<0 INTVL=1; end
% %               TMP = str2num(fgets(LINP));
% %               [DT,BETA,GAMA] = deal(TMP(1),TMP(2),TMP(3));
% %               if INTIAL~=0
% %                 TMP = str2num(fgets(LINP));
% %                 for I=1:NEQ GLU(I)=TMP(I); end
% %                 TMP = str2num(fgets(LINP));
% %                 for I=1:NEQ GLV(I)=TMP(I); end
% %               end
% %             else
% %               NVCTR=fscanf(LINP,'%e',1);
% %             end
% %           end

%-----
% OUTPUT : Writes the input data in to the output file
%-----
fprintf(LOUT,'\n \t \t Output for Input Data from file %s\n',FILE1);
fprintf(LOUT,'\n \t \t P. RAMESH \n\n');
if ITYPE<1
    fprintf(LOUT,'\t***** CLASSICAL LAMINATE PLATE THEORY
*****\n');
    if ITYPE==0 fprintf(LOUT,'\t***** Non-conforming plate element is
used *****\n'); end
    if ITYPE==1 fprintf(LOUT,'\t***** Conforming plate element is
used *****\n'); end
else
    fprintf(LOUT, '\t***** FIRST-ORDER SHEAR DEFORMATION THEORY
*****\n');
end
if IDYN==0
    fprintf(LOUT,'\t***** A STEADY-STATE PROBLEM is analyzed
*****\n');
else
    if NEIGN>0
        fprintf(LOUT,'\t***** An EIGENVALUE PROBLEM is analyzed
*****\n');
    else
        fprintf(LOUT, '\t***** A TRANSIENT PROBLEM is analyzed
*****\n');
    end
end
fprintf(LOUT,'\n \n%s\n',TITLE);
fprintf(LOUT,'Number of layers in the laminate .....= %4d
\n',LAYERS);
fprintf(LOUT,'The layer orientations and thicknesses are:(identical
material layers) \n ');
for I=1:LAYERS    fprintf(LOUT,'%4d,%4.4e\n',THETA(I),THKNS(I)); end
for M=1:NUMAT
    fprintf(LOUT,'\n P R O P E R T I E S of material no. = %d \n\n
',M);

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        fprintf(LOUT, 'Youngs modulus,   E1 ..... = %e
\n', E1(M));
        fprintf(LOUT, 'Youngs modulus,   E2 ..... = %e
\n', E2(M));
        fprintf(LOUT, 'Shear modulus,   G12 ..... = %e
\n', G12(M));
        fprintf(LOUT, 'Shear modulus,   G13 ..... = %e
\n', G13(M));
        fprintf(LOUT, 'Shear modulus,   G23 ..... = %e
\n', G23(M));
        fprintf(LOUT, 'Poissons ratio, NU12 ..... = %e
\n', ANU12(M));
end
if IDYN>0
    fprintf(LOUT, 'Density of the material, RHO ..... = %e
\n', RHO(M));
end
if NTHERM>0
    fprintf(LOUT, 'Coefficient of thermal expansion, ALFA1 = %e
\n', ALFA1(M));
    fprintf(LOUT, 'Coefficient of thermal expansion, ALFA2 = %e
\n', ALFA2(M));
end
if NTHERM>0
    fprintf(LOUT, 'Uniform temperature thru thickness, T0 = %e
\n', T0);
    fprintf(LOUT, 'Temperature gradient thru thickness, T1 = %e
\n', T1);
end
if NEIGN==0
    fprintf(LOUT, 'Intensity of transverse load, Q0 ..... = %e
\n', Q0);
    fprintf(LOUT, 'Linear part of transverse load, QX ..... = %e
\n', QX);
    fprintf(LOUT, 'Quadratic part of transverse load, QY .. = %e \n',
QY);
    if LOAD==2
        fprintf(LOUT, 'Sinusoidal distribution of the load is used \n');
        fprintf(LOUT, '(1 = quarter plate model; 2 = full plate = %e
\n', LPT);
    end
end

% % Dynamic load
% if IDYN~=0
%     if NEIGN==0
%         if INTVL<=0 INTVL=1; end
%         fprintf(LOUT, 'Time increment used, DT ..... = %e
\n', DT);
%         fprintf(LOUT, 'Parameter BETA in the Newmark scheme ... = %e
\n', BETA);
%         fprintf(LOUT, 'Parameter GAMA in the Newmark scheme ... = %e
\n', GAMA);
%         fprintf(LOUT, 'Number of time steps used, NTIME ..... = %e
\n', NTIME);
%         fprintf(LOUT, 'Time step at which load is removed, NSTP = %e
\n', NSTP);

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%      fprintf(LOUT,'Time interval at which soln. is printed = %e
\n', INTVL);
%      DT2=DT*DT;
%      A1=(1.0-BETA)*DT;
%      A2=BETA*DT;
%      A3=2.0/GAMA/DT2;
%      A4=A3*DT;
%      A5=1.0/GAMA-1.0;
%      if INTIAL>0
%          for I=1:NEQ GLA(I)=0.0; end
%      else
%          for I=1:NEQ
%              GLU(I)=0.0;
%              GLV(I)=0.0;
%              GLA(I)=0.0;
%          end
%      end
%  end
%  end

if ITYPE<=1
    NSHR=0;
else
    NSHR=1;
end
if IELTYP==0
    fprintf(LOUT, '\n \t *** A mesh of    TRIANGLES    is chosen by
user *** \n');
else
    fprintf(LOUT, '\n \t*** A mesh of QUADRILATERALS is chosen by user
*** \n');
end
%      Compute the half bandwidth of the global coefficient matrix
if NEIGN==0
    NHBW=0;
    for N=1:NEM
        for I=1:NPE
            for J=1:NPE
                NW=(abs(NOD(N,I)-NOD(N,J))+1)*NDF;
                if NHBW<NW NHBW=NW; end
            end
        end
    end
else
    NHBW=NEQ;
end

fprintf(LOUT, '\n \t FINITE ELEMENT MESH INFORMATION:\n \n');
fprintf(LOUT, 'Element type: 0 = Triangle; >0 = Quad.) = %d
\n', IELTYP);
fprintf(LOUT, 'Number of nodes per element, NPE ..... = %d \n', NPE);
fprintf(LOUT, 'No. of primary deg. of freedom/node, NDF = %d \n', NDF);
fprintf(LOUT, 'No. of deg. of freedom per element, NN = %d \n', NN);
fprintf(LOUT, 'Number of elements in the mesh, NEM .... = %d \n', NEM);
fprintf(LOUT, 'Number of nodes in the mesh, NNM ..... = %d \n', NNM);
fprintf(LOUT, 'Number of equations to be solved, NEQ .. = %d \n', NEQ);

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fprintf(LOUT, 'Half bandwidth of the matrix GLK, NHBW = %d \n', NHBW);
if MESH==1
    fprintf(LOUT, 'Mesh subdivisions, NX and NY ..... =%d \t %d
\n', NX, NY);
end
fprintf(LOUT, 'No. of specified generalized displ., NSPV= %d \n', NSPV);
if NSSV~=0
    fprintf(LOUT, 'No. of specified generalized forces, NSSV= %d
\n', NSSV);
    fprintf(LOUT, 'Node DOF Value\n');
    for IB=1:NSSV
        for JB=1:2
            fprintf(LOUT, '%d', ISSV(IB, JB));
        end
        fprintf(LOUT, '%d', VSSV(IB));
    end
end
fprintf(LOUT, '_____
\n');
fprintf(LOUT, '\t Node x-coord. y-coord. \t\t Specified
displacements and forces \n \t\t\t\t(0, unspecified; >0, specified)
Displ. DOF Force DOF\n');
fprintf(LOUT, '_____
\n');
for IM=1:NNM %140
    for K=1:NDF
        IBP(K)=0;
        IBS(K)=0;
    end
    if NSPV~=0
        for JP=1:NSPV
            NODE=ISPV(JP, 1);
            NDOF=ISPV(JP, 2);
            if NODE==IM
                IBP(NDOF)=NDOF;
            end
        end
    end
    if NSSV~=0
        for JS=1:NSSV
            NODE=ISSV(JS, 1);
            NDOF=ISSV(JS, 2);
            if NODE.EQ.IM
                IBS(NDOF)=NDOF;
            end
        end
    end
    if NDF==5
        fprintf(LOUT, '\n \t %d ', IM);
        for J=1:2 fprintf(LOUT, '\t %e', GLXY(IM, J)); end
        for K=1:NDF fprintf(LOUT, '\t %d', IBP(K)); end
        for K=1:NDF fprintf(LOUT, '\t %d', IBS(K)); end
    else
        fprintf(LOUT, '\n \t %d ', IM);
        for J=1:2 fprintf(LOUT, '\t %e', GLXY(IM, J)); end
        for K=1:NDF fprintf(LOUT, '\t %d', IBP(K)); end
        for K=1:NDF fprintf(LOUT, '\t %d', IBS(K)); end
    end
end

```



```

end
end
fprintf(LOUT, '\n
_____
_____ \n');
% Define the polynomial degree and number of integration points
% (based on the CONSTANT laminate stiffnesses, [A],[B],[D], etc.)
fprintf(LOUT, '\n \n \t NUMERICAL INTEGRATION DATA:\n\n');
fprintf(LOUT, 'Full quadrature (IPDF x IPDF) rule, IPDF = %d \n', IPDF);
fprintf(LOUT, 'Reduced quadrature (IPDR x IPDR), IPDR = %d \n', IPDR);
fprintf(LOUT, 'Quadrature rule used in postproc., ISTR = %d \n', ISTR);
if NPRNT==1
    fprintf(LOUT, '\nConnectivity Matrix, [NOD] \n');
    for I=1:NEM
        fprintf(LOUT, '\n \t %d ', I);
        for J=1:NPE fprintf(LOUT, '\t %d' ,NOD(I,J)); end
    end
end
end

% %Deformation Ploting -3D
% plotting
figure
for i=1:NEM
    for j=1:NPE
        xel(j)=GLXY(NOD(i,j),1);
        yel(j)=GLXY(NOD(i,j),2);
        zel=zeros(size(xel));
    end
    % plot3(xel,yel,zel);
    fill3(xel,yel,zel,'m');
    hold on
end

%-----
% * P R O C E S S O R U N I T *
%-----
% optloc=1;
% for P=1:NEM+1 %forms Bact
% I=1:NEM; PIEZO_E(I)=0;
% PIEZO_E(P)=1; %sets Piezo=1 for element-1 or location-1
% % if P>NEM PIEZO_E(optloc)=1; end
% if P>NEM PIEZO_E=PIEZO_E_in; end
PIEZO_E=PIEZO_E_in;
for NTK=1:NTHKNS %350
    HSCALE=DH(NTK);
    HT=0.0;
    for I=1:LAYERS
        HT=HT+THKNS(I)*HSCALE;
    end
    HT2=HT*HT;
    HT3=HT2*HT;
    YL2=YL*YL;
    YL3=YL*YL2;
    YL4=YL2*YL2;
    % SCALEU=100.0*E2(1)*HT2/Q0/YL3;
    % SCALEW=100.0*E2(1)*HT3/Q0/YL4;

```

```

fprintf(LOUT, '\n\nTotal thickness of the laminate, HT = %e \n', HT);

% Calculate laminate stiffnesses A,B,D and write them in to output
file
PLTSTF(LAYERS, NSHR, IDYN, N THERM, NPRNT, HSCALE);
if PIEZO>0
    [SP]=piezo_resultant(p_tk, p_theta, EC13, EC23, EC33, EC41,
EC42, EC51, EC52, AV, NPLY, ITYPE);
end

%     if IDYN>0 & NEIGN==0    TIME=0.0;     end
NT = 0;
NCOUNT=0;
%loop=1;
% while loop~=0
%     NCOUNT=NCOUNT+1; %160
%     if IDYN~=0 & NEIGN==0
%         if NCOUNT>=NSTP
%             Q0=0.0;
%             QX=0.0;
%             QY=0.0;
%         end
%     end
%     % Initialize the global coefficient matrices and vectors
GLF=zeros(1, NEQ);
%     GLK=zeros(NEQ, NHBW);
%     if NEIGN>0 GLM=zeros(NEQ, NHBW); end

%     Initialize the global coefficient matrices and vectors
GLFN=zeros(1, NEQ);
GLKN=zeros(NEQ, NEQ);
GLMN=zeros(NEQ, NEQ);

%Do-loop on the number of ELEMENTS to compute element matrices
%and their assembly begins here
%     disp(sprintf('E n t e r i n g..element loop'));
for N=1:NEM % 250
    for I=1:NPE
        NI=NOD(N, I);
        ELXY(I, 1)=GLXY(NI, 1);
        ELXY(I, 2)=GLXY(NI, 2);
%         if NEIGN==0
%             if IDYN>0
%                 LI=(NI-1)*NDF;
%                 L = (I-1)*NDF;
%                 for J=1:NDF
%                     LI=LI+1;
%                     L=L+1;
%                     % Initializing the displacement, velocity,
%                     % acceleration for dynamic analysis
%                     ELU(L)=GLU(LI);
%                     ELV(L)=GLV(LI);
%                     ELA(L)=GLA(LI);
%                 end
%             end
%         end
    end
end
end

```

end

```
ELKMFR_D_F(AK, XL, YL, LPT, NEIGN, NPE, NDF, ITYPE, IDYN, LOAD, N THERM, INTIAL, NCO  
UNT, N);
```

```
%      %Print element matrices and vectors (only when NPRNT=1 or  
NPRNT=3)  
%      if NCOUNT==1  
%          if NPRNT==1 | NPRNT==3  
%              if N==1 %for element no 1 only  
%                  fprintf(LOUT, '\n\n \t Element stiffness matrix:  
%d Element \n', N);  
%                  for I=1:NN  
%                      fprintf(LOUT, '\n');  
%                      for J=1:NN  
%                          fprintf(LOUT, '\t %e', ELK(I, J));  
%                      end  
%                  end  
%                  if NEIGN==0  
%                      fprintf(LOUT, '\n\n \t Element force  
vector:\n')  
%                      fprintf(LOUT, '\n');  
%                      for J=1:NN  
%                          fprintf(LOUT, '\t %e', ELF(J));  
%                      end  
%                  else  
%                      fprintf(LOUT, '\n\n Element mass matrix:  
\n');  
%                      for I=1:NN  
%                          fprintf(LOUT, '\n');  
%                          for J=1:NN  
%                              fprintf(LOUT, '\t %e', ELM(I, J));  
%                          end  
%                      end  
%                  end  
%              end  
%          end  
%      end  
end
```

```
[index]=feeldof(NOD, NPE, NDF, N);  
[GLKN, GLFN]=feasmb12(GLKN, GLFN, index);  
[GLMN]=feasmb11(GLMN, index);
```

```
for I=1:NPE  
%     NR=(NOD(N, I)-1)*NDF;  
%     for II=1:NDF  
%         NR=NR+1;  
%         L=(I-1)*NDF+II;  
%         if NEIGN==0  
%             GLF(NR)=GLF(NR)+ELF(L);  
%         end  
%     for J=1:NPE  
%         if NEIGN==0  
%             NCL=(NOD(N, J)-1)*NDF;  
%         else
```

```

%
%          NC=(NOD(N,J)-1)*NDF;
%
%          end
%          for JJ=1:NDF
%              M=(J-1)*NDF+JJ;
%              if NEIGN==0
%                  NC=NCL+JJ+1-NR;
%                  if NC>0
%                      GLK(NR,NC)=GLK(NR,NC)+ELK(L,M);
%                  end
%              else
%                  NC=NC+1;
%                  GLK(NR,NC)=GLK(NR,NC)+ELK(L,M);
%                  GLM(NR,NC)=GLM(NR,NC)+ELM(L,M);
%              end
%          end %240
%      end %240
%  end%240
%end%240

end % 250 (ELEMENT LOOP ends)

%
%   %Print global matrices when NPRNT > 2
%   if NCOUNT<=1
%       if NPRNT>=2
%           fprintf(LOUT, '\n Global stiffness matrix (upper band):
\n');
%           for I=1:NEQ
%               fprintf(LOUT, '\n');
%               for J=1:NHBW
%                   fprintf(LOUT, '\t %e', GLK(I,J));
%               end
%           end
%           if NEIGN==0
%               fprintf(LOUT, '\n Global force vector:\n');
%               for I=1:NHBW
%                   fprintf(LOUT, '\t %e', GLF(I));
%               end
%           else
%               fprintf(LOUT, '\n Global mass matrix (full form):\n
');
%               for I=1:NEQ
%                   for J=1:NHBW
%                       fprintf(LOUT, '\t %e', GLM(I,J));
%                   end
%               end
%           end
%       end
%   end
%   end
%   if NEIGN>0 commneted for static analysis..
%   EGNBOU(GLK, GLM, IBDY, ISPV, MAXSPV, NDF, NEQ, NEQR, NSPV, NRMAX)
%   % AXLBX; % CALL AXLBX
%   (NEQR, GLK, GLM, EGNVAL, EGNVEC, JVEC, NROT, NRMAX)
%   % fprintf(LOUT, '\n S O L U T I O N :\n ');
%   % fprintf(LOUT, '\n Number of Jacobi iterations ..... NROT =
%d\n', NROT);
%   % CONST=(YL*YL/HT)*DSQRT(RHO(1)/E2(1));

```

```

% for I=1:NEQR
% if IDYN>0 & NEIGN==1
% VALUE = DSQRT(EGNVAL(I))*CONST;
%   fprintf(LOUT, '\n Eigenvalue(%d) =%e \t\t Frequency =
%e\n', I, EGNVAL(I), VALUE);
%   else
%       fprintf(LOUT, 'Buckling load(%d)
=%e\n', I, EGNVAL(I));
%       end
%       if NVCTR~=0
%           fprintf(LOUT, '\n Eigenvector:\n');
%           for J=1:NEQR fprintf(LOUT, '\t\t %e\n', EGNVEC(J,I));
end
%       end
%       end
%       STOP

%else %NEIGN CONDITION :(
%   [GLKE, GLME]=feaplycs(GLKN, GLMN, ISPV, NSPV, NEQ);
%   fsol=eig(GLKE, GLME);
%   fsol=sqrt(fsol);
%   num=1:1:NEQ;
%   frequency=[num' fsol]

% Static Analysis
GLK1=GLKN; %before applying BC
GLM=GLMN; %before applying BC
[GLKN, GLFN]=feaplyc2(GLKN, GLFN, ISPV, ISSV, NDF, NEQ, NSPV, NSSV, VSPV, VSSV);
[GLKE, GLME]=feaplycs(GLK1, GLMN, ISPV, NSPV, NEQ);
IRES=0;
FORCE=GLFN;
SP_GLKN=sparse(GLKN);
GLKN_UP=chol(SP_GLKN);
GLFN=GLKN_UP\ (GLKN_UP'\GLFN');

% Dynamic Analysis
% if IDYN>0
%
%   For nonzero initial conditions, GLF in the very first
solution
%   is the acceleration, {A}=[MINV]({F}-[K]{U})
%
%       if NCOUNT==1 & INTIAL~=0
%           if IDYN>0
%               for I=1:NEQ
%                   GLA(I)=GLF(I);
%               end
%               fprintf(LOUT, '\n *TIME* = %e (Initial
acceleration vector:)\n');
%               for I=1:NEQ fprintf(LOUT, '\t %e', GLA(I)); end
%           end
%       end
%   end

```

```

%           else
%           NT = NT + 1;
%           TIME=TIME+DT;
%       end
%       else
%           NT = NT + 1;
%           TIME=TIME+DT;
%       end
%       for I=1:NEQ
%           GLU(I)=A3*(GLF(I)-GLU(I))-A4*GLV(I)-A5*GLA(I);
%           GLV(I)=GLV(I)+A1*GLA(I)+A2*GLU(I);
%           GLA(I)=GLU(I);
%           GLU(I)=GLF(I);
%       end
%       INTGR=(NT/INTVL)*INTVL;
%       if INTGR==NT
%           NFLAG=1;
%       else
%           NFLAG=0;
%       end
%       else%IDYN>0 condition
%       for I=1:NEQ
%           GLU(I)=GLF(I);
%       end
%       %end %IDYN>0 condition
%       Print the solution (i.e., nodal values of the primary
variables)
%       if IDYN>0
%           fprintf(LOUT,'\n*TIME* = %e\t Time Step Number
=%d',TIME,NT);
%       end
%       disp(sprintf('E n t e r i n g... soln..'));

%       if P==NEM
%           NFLAG=1;
%       else
%           NFLAG=0;
%       end

NFLAG=1;

if NFLAG>0
    fprintf(LOUT, '\n\n\n\t\tS O L U T I O N :\n');
    %           displacements
%
fprintf(LOUT, '\n_____
\n');
%           if NDF<=5 fprintf(LOUT, 'Node x-coord. y-coord.
displ. u displ. v deflec. w rot. dw/dx rot. dw/dy\n'); end
%           if NDF>5 fprintf(LOUT, 'Node x-coord. y-coord.
displ. u displ. v deflec. w rot. phi-x rot. phi-y\n'); end
%
fprintf(LOUT, '\n_____
\n');
%           NDF1=5;
%           for I=1:NNM % 320

```

```

%           II=NDF*(I-1)+1;
%           JJ=II+NDF1-1;
%           GLF(II)=GLF(II)*SCALEU;
%           GLF(II+1)=GLF(II+1)*SCALEU;
%           GLF(II+2)=GLF(II+2)*SCALEW;
%           fprintf(LOUT, '\n %d \t', I);
%           for J=1:2 fprintf(LOUT, '\t %e\t', GLXY(I, J)); end
%           for J=II:JJ fprintf(LOUT, '%e\t', GLF(J)); end
%       end % 320 ends...
%
fprintf(LOUT, '\n_____
_____ \n');
%

fprintf(LOUT, '\n\n\n\t\t NEW S O L U T I O N :\n');
%           displacements

fprintf(LOUT, '\n_____
_____ \n');
%           if NDF<=5 fprintf(LOUT, 'Node   x-coord.   y-coord.   displ.
u displ. v   deflec. w   rot. dw/dx   rot. dw/dy\n'); end
%           if NDF>5 fprintf(LOUT, 'Node   x-coord.   y-coord.   displ.
u displ. v   deflec. w   rot. phi-x   rot. phi-y\n'); end

fprintf(LOUT, '\n_____
_____ \n');
%           NDF1=5;
%           for I=1:NNM % 320
%               II=NDF*(I-1)+1;
%               JJ=II+NDF1-1;
%               GLFN(II)=GLFN(II)*SCALEU;
%               GLFN(II+1)=GLFN(II+1)*SCALEU;
%               GLFN(II+2)=GLFN(II+2)*SCALEW;
%               GLFN(II)=GLFN(II);
%               GLFN(II+1)=GLFN(II+1);
%               GLFN(II+2)=GLFN(II+2);
%               fprintf(LOUT, '\n %d \t', I);
%               for J=1:2 fprintf(LOUT, '\t %e\t', GLXY(I, J)); end
%               for J=II:JJ fprintf(LOUT, '%e\t', GLFN(J)); end
%           end % 320 ends...

fprintf(LOUT, '\n_____
_____ \n');
%           %Newly added ends

%           if NDF>NDF1
%               fprintf(LOUT, '\n Nodal values of W,xy for conforming plate
element:\n ');
%           end
%           if IGRAD~=0
%               % * P O S T P R O C E S S O R   U N I T   *
%               fprintf(LOUT, '\n\n STRESSES AT top (T) and bottom (B) of
each layer are printed below: \n');

```

```

%
fprintf(LOUT, '\n\t\t_____
\n');
%
fprintf(LOUT, '\n\t Ply sgmxx-T sgmxx-B sggmyy-T
sgmyy-B sgmxy-T sgmxy-B sgmxz-T sgmyz-B\n');
%
fprintf(LOUT, '\n\t\t_____
\n');
%
% Compute the stresses
%
% ICOUNT=1;
% for N=1:NEM
% for I=1:NPE
% NI=NOD(N, I);
% ELXY(I, 1)=GLXY(NI, 1);
% ELXY(I, 2)=GLXY(NI, 2);
% LI=(NI-1)*NDF;
% L=(I-1)*NDF;
% for J=1:NDF
% LI=LI+1;
% L=L+1;
% ELU(L)=GLU(LI);
% end
% end
% disp(sprintf('\t\tcalling STRESS for elemnt
%d..', N));
% STRESS; % CALL
STRESS(ELU, ELXY, HT, Q0, YL, ISTR, ITYPE, LAYERS, NDF, NPE, NTHERM, ICOUNT, MAXELM
, MAXGPT, RNXX, RNYY, RNXY, RMXX, RMYX, RMY, RQX, RQY)
% ICOUNT=ICOUNT+1 ;
% end %element loop for STRESS ends...
% fprintf(LOUT, '\n_____ : )
THE END :) _____\n');
% end % IGRAD condition...
%
% check for IDYN>0 is modified to avoid GOTO and is modified
with WHILE
% new control variable "loop" has been used..
% if IDYN>0
% if NT>=NTIME
% loop=0;
% else
% loop=1;
% end
% end
% end
% end
% end

% end%while loop ends here.....
end % NFLAG condition
end % NTK loop ends...

% % % Forming Bact
% if P<=NEM
% N=NEQ;
% F=FORCE';
% [V, D]=eig(K, M);

```



```

% FF=V'*F;
%
% F_vec = [zeros(size(FF))
%          inv(M)*FF];
% Bact(:,P)=[F_vec];
% end

% % % First order System
% if P==NEM
% K=GLKN;
% M=GLME;
% [V,D]=eig(K,M);
% KK=V'*K*V;
% MM=V'*M*V;
%
% C=0.2*MM+0.005*KK; % Alpha=0.2, Beta=0.005
% A_mat = [zeros(size(KK)) eye(size(KK))
%          -inv(MM)*KK      -inv(MM)*C];
% C_mat=[eye(size(A_mat)/2), zeros(size(A_mat)/2)];
% end

% Optimal location calculation
% if P>NEM
% [optloc]=optactuator(A_mat,Bact,C,10,0.001,100,1,0);
%optactuator(A_mat,Bact,C,optact0,tol,itmax,type,cov);
% % % %Hand calculation
% % Q=C_mat'*C_mat;
% % R=1;
% % for j=1:NEM
% %     B_mat=Bact(:,j);
% %     [f,K]=lqr(A_mat,B_mat,Q,R);
% %     [Vs,es]=eig(K);
% %     sigmahand(j)=.5*max(diag(es));
% % end;
% % ophand=min(sigmahand);
% % optacthand=find(sigmahand<opthand+.00001)
% end

% disp(sprintf(' loaction %d',P))

% end % Bact loop ends
disp(sprintf('Static analysis compleeted...'));
fclose('all');
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%% end of main program %%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%

```

```

% %-----
% % Dynamic Analysis
% %-----
% t0 = 0;
% t1 = 2.5;
% nsteps1 = round((t1-t0)*100);
% hsteps1 = (t1-t0)/nsteps1;

```

```

% N=NEQ;
% K=GLKN;
% M=GLME;
% F=FORCE';
% %-----
% %Eigen value analysis
% % %-----
% % freq=eig(K,M);
% % freq=sqrt(freq)
% % %-----
%
% C=0.2*M+0.005*K; % Alpha=0.2, Beta=0.005
% A_mat = [zeros(size(K)) eye(size(K))
%          -inv(M)*K      -inv(M)*C];
% % A_mat = [zeros(size(K)) eye(size(K))
% %          -inv(M)*K      eye(size(K))];
% F_vec = [zeros(size(F))
%          inv(M)*F];
% %
% % Integrate system using the stiff ODE solver odel5s.m.
% %
% y0 = zeros(2*N,1);
% tspan = t0:hsteps1:t1;
% options =
odeset('JConstant','on','Jacobian','on','BDF','on','RelTol',1e-4,
'AbsTol',1e-4);
% [t,y] = ode23s('yprime_plate',tspan,y0,options,A_mat,F_vec);
% yt1 = y(:,1:N);
% % ysolution = Bvn*yt1';
% ysolution = yt1';
%
% figure(1)
% plot(t,ysolution(((NNM-1)/2)*5+3,:))
% xlabel('Time (s)')
% ylabel('Displacement ( )')
% %

%Deformation Ploting -3D
J=1;
for I=3:5:NNM*5
    Z(J)=GLFN(I);
    J=J+1;
end
% [XP,YP]=meshgrid(GLXY(:,1),GLXY(:,2));
% ZP=griddata(GLXY(:,1),GLXY(:,2),Z,XP,YP);
% figure
% surf(XP,YP,ZP);

% % plotting
figure
for i=1:NEM
    for j=1:NPE
        xel(j)=GLXY(NOD(i,j),1);
        yel(j)=GLXY(NOD(i,j),2);
        zel(j)=Z(NOD(i,j));
    end
end

```

```

%     plot3(xel,yel,zel);
    fill3(xel,yel,zel,'g');
    hold on
end

elapsed_time=cputime-stsrt_time

% %Linear System simulation
% t = 0:1/200:10;
% sys = ss(A_mat, Bact(:,20), C_mat,0);
% [y,t] = lsim(sys,60*sin(t),t);
% figure(2)
% plot(t,y(:,end-1),'r-.');
% xlabel('Time [s]');
% ylabel('Amplitude [m]');

% %Step response
% figure(3)
% y=step(A_mat,Bact(:,20),C_mat,0,1,t);
% plot(t,y(:,1))

```

#### **aplybcdyn.m**

```

function [kkk,mmm,fff]=aplybcdyn(bcdof, kk, mm, ff)
%Apply BC for dynamic analysis - general way
kkk=kk;
mmm=mm;
fff=ff;
% Row
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpk = [kkk(bcdof(i)-i+2:end,:)];
        tmpm = [mmm(bcdof(i)-i+2:end,:)];
    elseif bcdof(i)-i+1 == size(kkk,1)
        tmpk = [kkk(1:bcdof(i)-i,:)];
        tmpm = [mmm(1:bcdof(i)-i,:)];
    else
        tmpk = [kkk(1:bcdof(i)-i,:); kkk(bcdof(i)-i+2:end,:)];
        tmpm = [mmm(1:bcdof(i)-i,:); mmm(bcdof(i)-i+2:end,:)];
    end
    kkk = tmpk;
    mmm = tmpm;
end
% Col
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpk = [kkk(:,bcdof(i)-i+2:end)];
        tmpm = [mmm(:,bcdof(i)-i+2:end)];
    elseif bcdof(i)-i+1 == size(kkk,2)
        tmpk = [kkk(:,1:bcdof(i)-i)];
        tmpm = [mmm(:,1:bcdof(i)-i)];
    else
        tmpk = [kkk(:,1:bcdof(i)-i) kkk(:,bcdof(i)-i+2:end)];
        tmpm = [mmm(:,1:bcdof(i)-i) mmm(:,bcdof(i)-i+2:end)];
    end
    kkk = tmpk;

```

```

    mmm = tmpm;
end
% F
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpf = [fff(bcdof(i)-i+2:end)];
    elseif bcdof(i)-i+1 == length(fff)
        tmpf = [fff(1:bcdof(i)-i)];
    else
        tmpf = [fff(1:bcdof(i)-i); fff(bcdof(i)-i+2:end)];
    end
    fff = tmpf;
end
fff=fff';

bcgen.m
function
[NSPV, ISPV, VSPV]=bcgen(NOD, GLXY, NNM, NEM, bcx0, bcxl, bcy0, bcyl, XL, YL)
nx0=0; ny0=0; nxl=0; nyl=0;
% BGLXY=GLXY*1000; BXL=XL*1000; BYL=YL*1000; % Dummy multipliers
err=10e-12;
for i=1:NNM
    if GLXY(i,1)<=err
        nx0=nx0+1;
        edgx0(nx0)=i;
    end
    if GLXY(i,2)<=err
        ny0=ny0+1;
        edgy0(ny0)=i;
    end
    if (XL-GLXY(i,1))<=err
        nxl=nxl+1;
        edgxl(nxl)=i;
    end
    if (YL-GLXY(i,2))<=err
        nyl=nyl+1;
        edgyl(nyl)=i;
    end
end
iNSPV=0;
for i=1:NNM
    inx0=(edgx0==i); inxl=(edgxl==i);
    iny0=(edgy0==i); inyl=(edgyl==i);
    %if the node is at the corner
    if (sum(inx0)>0 & sum(iny0)>0) | (sum(inx0)>0 & sum(inyl)>0) |
    (sum(inxl)>0 & sum(iny0)>0) | (sum(inxl)>0 & sum(inyl)>0)
        ISPV(iNSPV+1,1)=i; ISPV(iNSPV+2,1)=i; ISPV(iNSPV+3,1)=i;
        ISPV(iNSPV+4,1)=i; ISPV(iNSPV+5,1)=i;
        if bcx0==1 %SS
            ISPV(iNSPV+1,2)=1; ISPV(iNSPV+2,2)=2; ISPV(iNSPV+3,2)=3;
        ISPV(iNSPV+4,2)=4; ISPV(iNSPV+5,2)=5;
        end
        iNSPV=iNSPV+5;
    elseif sum(inx0)>0 |sum(inxl)>0 %if the node is @x=0 OR x=L edges
        ISPV(iNSPV+1,1)=i; ISPV(iNSPV+2,1)=i; ISPV(iNSPV+3,1)=i;
        if bcx0==1 %SS
            ISPV(iNSPV+1,2)=2; ISPV(iNSPV+2,2)=3; ISPV(iNSPV+3,2)=5;

```

```

        end
        iNSPV=iNSPV+3;
    elseif sum(iny0)>0 |sum(iny1)>0 %if the node is @y=0 OR y=L edges
        ISPV(iNSPV+1,1)=i; ISPV(iNSPV+2,1)=i; ISPV(iNSPV+3,1)=i;
        if bcy0==1 %SS
            ISPV(iNSPV+1,2)=1; ISPV(iNSPV+2,2)=3; ISPV(iNSPV+3,2)=4;
        end
        iNSPV=iNSPV+3;
    end
end
NSPV=length(ISPV);
VSPV=zeros(NSPV,1); %zero bc values

```

### **BOUNRY.m**

```

function
[GLK, GLF]=BOUNRY (ISPV, ISSV, NDF, NEQ, NHBW, NSPV, NSSV, GLK, GLF, VSPV, VSSV, NCO
UNT, INTIAL)
%
%-----
%   The subroutine implements specified values of the primary and
%   secondary variables by modifying the coefficient matrix [S] and
%   (banded and symmetric) and the right-hand side vector {SL}.
%-----
%
S=GLK;
SL=GLF;
if NSSV~=0
    if INTIAL==0 | NCOUNT~=1
%
%   Implement specified values of the SECONDARY ARIABLES:_____
%
        for I=1:NSSV
            II=(ISSV(I,1)-1)*NDF+ISSV(I,2);
            SL(II)=SL(II)+VSSV(I);
        end
    end
end
%
%   Implement specified values of the PRIMARY VARIABLES
if NSPV~=0
    for NB=1:NSPV % loop 50 ..
        IE=(ISPV(NB,1)-1)*NDF+ISPV(NB,2);
        VALUE=VSPV(NB);
        IT=NHBW-1;
        I=IE-NHBW;
        for II=1:IT
            I=I+1;
            if I>1
                J=IE-I+1;
                SL(I)=SL(I)-S(I,J)*VALUE;
                S(I,J)=0.0;
            end
        end
    end
    S(IE,1)=1.0;
    SL(IE)=VALUE;
    I=IE;
    for II=2:NHBW % loop 40..

```

```

        I=I+1;
        if I<=NEQ
            SL(I)=SL(I)-S(IE,II)*VALUE;
            S(IE,II)=0.0;
        end
    end
end
end

GLK=S;
GLF=SL;

ELKMFR_D_F.m
function
[]=ELKMFR_D_F(AK,XL,YL,LPT,NEIGN,NPE,NDF,ITYPE,IDYN,LOAD,NTHERM,INTIAL,
NCOUNT,N)
global ELF ELK ELM ELXY ELU ELV ELA A1 A2 A3 A4 A5 Q0 QX QY      %
Stiff1
global IPDF IPDR NIPF NIPR                                     %
Point
global P0 P1 P2 P3 P4 P5 P6                                  %Integ
global T0 T1 TN TM TL TK                                    %Therm
global SF GDSF SFH GDSFH GDDSFH                             %Shape
global A B D E F G H A44 A45 A55 D44 D45 D55                %
Stiff2
global PIEZO_E PM PN

    GAUSPT=[0.0 0.0 0.0 0.0 0.0;
            -0.57735027 0.57735027 0.0 0.0 0.0;
            -0.77459667 0.0 0.77459667 0.0 0.0;
            -0.86113631 -0.33998104 0.33998104 0.86113631 0.0;
            -0.90617984 -0.53846931 0.0 0.53846931 0.9061798];
    GAUSPT=GAUSPT';
%

    GAUSWT=[2.0 0.0 0.0 0.0 0.0;
            1.0 1.0 0.0 0.0 0.0;
            0.55555555 0.88888888 0.55555555 0.0 0.0;
            0.34785485 0.65214515 0.65214515 0.34785485 0.0;
            0.23692688 0.47862867 0.56888888 0.47862867 0.23692688];
    GAUSWT=GAUSWT';

%
NN=NDF*NPE;
if ITYPE==0
    NET = 3;
else
    NET = 4;
end

ELK=zeros(45);
ELF=zeros(45,1);
ELM=zeros(45);
% Do-loops on numerical (Gauss) integration begin here. Subroutine
% SHPRCT (SHaPe functions for ReCTangular elements) is called here

```

```

for NI = 1:IPDF
for NJ = 1:IPDF
XI = GAUSPT(NI,IPDF);
ETA = GAUSPT(NJ,IPDF);
%
[DET]=SHPRCT(NPE,XI,ETA,ELXY,ITYPE);
%
CNST = DET*GAUSWT(NI,IPDF)*GAUSWT(NJ,IPDF);
X=0.0;
Y=0.0;
for I=1:NPE
X=X+ELXY(I,1)*SF(I);
Y=Y+ELXY(I,2)*SF(I);
end
%
if NEIGN==0
if LOAD==2
if LPT==1
QL=Q0*cos(pi*X/XL)*cos(pi*Y/YL);
else
QL=Q0*sin(pi*X/XL)*sin(pi*Y/YL);
end
else
QL=Q0+QX*X+QY*Y;
end
end
%
II=1;
for I=1:NPE
II1=II+1;
II2=II+2;
II3=II+3;
II4=II+4;
JJ=1;
for J=1:NPE
JJ1=JJ+1;
JJ2=JJ+2;
JJ3=JJ+3;
JJ4=JJ+4;
SXX=GDSF(1,I)*GDSF(1,J);
SXY=GDSF(1,I)*GDSF(2,J);
SYX=GDSF(2,I)*GDSF(1,J);
SYY=GDSF(2,I)*GDSF(2,J);
%
% All theories; stiffness coefficients for inplane
DOF:_____
%
ELK(II,JJ) =ELK(II,JJ) +CNST*(A(1,1)*SXX+A(1,3)*(SXY+SYX)
+A(3,3)*SYY);
ELK(II,JJ1) =ELK(II,JJ1)
+CNST*(A(1,2)*SXY+A(1,3)*SXX+A(2,3)*SYY+A(3,3)*SYX);
ELK(II1,JJ) =ELK(II1,JJ)
+CNST*(A(1,2)*SYX+A(1,3)*SXX+A(2,3)*SYY+A(3,3)*SXY);
ELK(II1,JJ1)=ELK(II1,JJ1)+CNST*(A(3,3)*SXX+A(2,3)*(SXY+SYX)+A(2,2)*SYY)
;
if IDYN>0

```

```

        S00=SF(I)*SF(J)*CNST;
        ELM(II,JJ) =ELM(II,JJ) +P0*S00;
        ELM(II1,JJ1)=ELM(II1,JJ1)+P0*S00;
    end
%
%      First-order theory; stiffness coefficients for bending
DOF:_____
%
    if ITYPE>1
        ELK(II,JJ3) =ELK(II,JJ3)
+CNST*(B(1,1)*SXX+B(1,3)*(SXY+SYX)+B(3,3)*SYY);
        ELK(II3,JJ) =ELK(II3,JJ)
+CNST*(B(1,1)*SXX+B(1,3)*(SXY+SYX)+B(3,3)*SYY);
        ELK(II,JJ4) =ELK(II,JJ4)
+CNST*(B(1,2)*SXY+B(1,3)*SXX+B(2,3)*SYY+B(3,3)*SYX);
        ELK(II4,JJ) =ELK(II4,JJ)
+CNST*(B(1,2)*SYX+B(1,3)*SXX+B(2,3)*SYY+B(3,3)*SXY);

        ELK(II1,JJ3)=ELK(II1,JJ3)+CNST*(B(1,3)*SXX+B(3,3)*SXY+B(1,2)*SYX+B(2,3)
*SYY);

        ELK(II3,JJ1)=ELK(II3,JJ1)+CNST*(B(1,3)*SXX+B(3,3)*SYX+B(1,2)*SXY+B(2,3)
*SYY);

        ELK(II1,JJ4)=ELK(II1,JJ4)+CNST*(B(3,3)*SXX+B(2,3)*(SXY+SYX)+B(2,2)*SYY)
;

        ELK(II4,JJ1)=ELK(II4,JJ1)+CNST*(B(3,3)*SXX+B(2,3)*(SXY+SYX)+B(2,2)*SYY)
;

        ELK(II3,JJ3)=ELK(II3,JJ3)+CNST*(D(1,1)*SXX+D(1,3)*(SXY+SYX)+D(3,3)*SYY)
;

        ELK(II3,JJ4)=ELK(II3,JJ4)+CNST*(D(1,2)*SXY+D(3,3)*SYX+D(1,3)*SXX+D(2,3)
*SYY);

        ELK(II4,JJ3)=ELK(II4,JJ3)+CNST*(D(1,2)*SYX+D(3,3)*SXY+D(1,3)*SXX+D(2,3)
*SYY);

        ELK(II4,JJ4)=ELK(II4,JJ4)+CNST*(D(3,3)*SXX+D(2,3)*(SXY+SYX)+D(2,2)*SYY)
;

        if IDYN>0
            ELM(II,JJ3) =ELM(II,JJ3) +P1*S00;
            ELM(II3,JJ) =ELM(II3,JJ) +P1*S00;
            ELM(II1,JJ4)=ELM(II1,JJ4)+P1*S00;
            ELM(II4,JJ1)=ELM(II4,JJ1)+P1*S00;
            ELM(II2,JJ2)=ELM(II2,JJ2)+P0*S00;
            ELM(II3,JJ3)=ELM(II3,JJ3)+P2*S00;
            ELM(II4,JJ4)=ELM(II4,JJ4)+P2*S00;
        end
    else

        for K=1:NET
            K1=(J-1)*NET+K;
            JK2=JJ2+K-1;

```



```

RXXX=GDSF(1,I)*GDDSFH(1,K1);
RXXY=GDSF(1,I)*GDDSFH(3,K1);
RXYI=GDSF(1,I)*GDDSFH(2,K1);
RYXX=GDSF(2,I)*GDDSFH(1,K1);
RYXI=GDSF(2,I)*GDDSFH(3,K1);
RYII=GDSF(2,I)*GDDSFH(2,K1);
ELK(II,JK2) =ELK(II,JK2) -
CNST*(B(1,1)*RXXX+B(1,2)*RXYI+2.0*B(2,3)*RXXY+B(1,3)*RYXX+B(2,3)*RYII+2
.0*B(3,3)*RXYI);
ELK(II1,JK2)=ELK(II1,JK2)-
CNST*(B(1,3)*RXXX+B(2,3)*RXYI+2.0*B(3,3)*RXXY+B(1,2)*RYXX+B(2,2)*RYII+2
.0*B(2,3)*RXYI);
if IDYN>0
SX0=GDSF(1,I)*SFH(K1);
SY0=GDSF(2,I)*SFH(K1);
ELM(II,JK2) =ELM(II,JK2) +P1*SX0*CNST;
ELM(II1,JK2) =ELM(II1,JK2) +P1*SY0*CNST;
end

end

for K=1:NET
K1=(I-1)*NET+K;
IK2=II2+K-1;
RXXX=GDSF(1,J)*GDDSFH(1,K1);
RXXY=GDSF(1,J)*GDDSFH(3,K1);
RXYI=GDSF(1,J)*GDDSFH(2,K1);
RYXX=GDSF(2,J)*GDDSFH(1,K1);
RYXI=GDSF(2,J)*GDDSFH(3,K1);
RYII=GDSF(2,J)*GDDSFH(2,K1);
ELK(IK2,JK2) =ELK(IK2,JK2) -
CNST*(B(1,1)*RXXX+B(1,2)*RXYI+2.0*B(2,3)*RXXY+B(1,3)*RYXX+B(2,3)*RYII+2
.0*B(3,3)*RXYI);
ELK(IK2,JK1)=ELK(IK2,JK1)-
CNST*(B(1,3)*RXXX+B(2,3)*RXYI+2.0*B(3,3)*RXXY+B(1,2)*RYXX+B(2,2)*RYII+2
.0*B(2,3)*RXYI);
if IDYN>0
SX0=GDSF(1,J)*SFH(K1);
SY0=GDSF(2,J)*SFH(K1);
ELM(IK2,JK2) =ELM(IK2,JK2) +P1*SX0*CNST;
ELM(IK2,JK1) =ELM(IK2,JK1) +P1*SY0*CNST;
end

end

end

for K=1:NET
K1=(I-1)*NET+K;
IK2=II2+K-1;
for L=1:NET
L1=(J-1)*NET+L;
JL2=JJ2+L-1;
TXXX=GDDSFH(1,K1)*GDDSFH(1,L1);
TXXY=GDDSFH(1,K1)*GDDSFH(2,L1);
TXXI=GDDSFH(1,K1)*GDDSFH(3,L1);
TXIX=GDDSFH(3,K1)*GDDSFH(1,L1);
TYYIX=GDDSFH(2,K1)*GDDSFH(1,L1);

```

```

        TYYXY=GDDSFH(2,K1)*GDDSFH(3,L1);
        TXYYY=GDDSFH(3,K1)*GDDSFH(2,L1);
        TYYYY=GDDSFH(2,K1)*GDDSFH(2,L1);
        TXYXY=GDDSFH(3,K1)*GDDSFH(3,L1);

ELK(IK2,JL2)=ELK(IK2,JL2)+CNST*(D(1,1)*TXXXX+D(1,2)*(TXXYY+TYYXX)+2.0*D
(1,3)*(TXXXY+TXYYX)+2.0*D(2,3)*(TXYYY+TYYXY)+4.0*D(3,3)*TXYXY+D(2,2)*TY
YYY);
        if IDYN>0
            S00=SFH(K1)*SFH(L1);
            SXY=GDSFH(1,K1)*GDSFH(1,L1)+GDSFH(2,K1)*GDSFH(2,L1);
            ELM(IK2,JL2) =ELM(IK2,JL2)+CNST*(P0*S00+P2*SXY);
        end
    end
end
end
    JJ = NDF*J+1;
end

if ITYPE>1
    ELF(II2)=ELF(II2)+CNST*SF(I)*QL;
    if N THERM~=0
        ELF(II) =ELF(II) -CNST*(GDSF(1,I)*TN(1)+GDSF(2,I)*TN(3));
        ELF(II1)=ELF(II1)-CNST*(GDSF(1,I)*TN(3)+GDSF(2,I)*TN(2));
        ELF(II3)=ELF(II3)-CNST*(GDSF(1,I)*TM(1)+GDSF(2,I)*TM(3));
        ELF(II4)=ELF(II4)-CNST*(GDSF(1,I)*TM(3)+GDSF(2,I)*TM(2));
    end
    if PIEZO_E(N)~=0
        ELF(II) =ELF(II) -CNST*(GDSF(1,I)*PN(1)+GDSF(2,I)*PN(3));
        ELF(II1)=ELF(II1)-CNST*(GDSF(1,I)*PN(3)+GDSF(2,I)*PN(2));
        ELF(II3)=ELF(II3)-CNST*(GDSF(1,I)*PM(1)+GDSF(2,I)*PM(3));
        ELF(II4)=ELF(II4)-CNST*(GDSF(1,I)*PM(3)+GDSF(2,I)*PM(2));
    end
else
    for K=1:NET
        IK2=II2+K-1;
        K1=(I-1)*NET+K;
        ELF(IK2)=ELF(IK2)+CNST*SFH(K1)*QL;
        if N THERM~=0

ELF(IK2)=ELF(IK2)+CNST*(GDDSFH(1,K1)*TM(1)+2.0*GDDSFH(3,K1)*TM(3)+GDDSF
H(2,K1)*TM(2));
            end
            if PIEZO_E(N)~=0

ELF(IK2)=ELF(IK2)+CNST*(GDDSFH(1,K1)*PM(1)+2.0*GDDSFH(3,K1)*PM(3)+GDDSF
H(2,K1)*PM(2));
            end
            end
        end
        II = NDF*I+1;
    end
end
end
end
%
% First-order theory; stiffness coefficients for transverse shear.
% Use reduced integration to evaluate coefficients associated with

```

```

%      TRANSVERSE SHEAR
terms:_____
%
%      if ITYPE>1
%      for NI=1:IPDR
%      for NJ=1:IPDR
%      XI = GAUSPT(NI, IPDR);
%      ETA = GAUSPT(NJ, IPDR);
%
%      [DET]=SHPRCT(NPE, XI, ETA, ELXY, ITYPE);
%
%      CNST=DET*GAUSWT(NI, IPDR)*GAUSWT(NJ, IPDR)*AK;
%
%      II=1;
%      for I=1:NPE
%      II2=II+2;
%      II3=II+3;
%      II4=II+4;
%      JJ = 1;
%      for J=1:NPE
%      JJ2=JJ+2;
%      JJ3=JJ+3;
%      JJ4=JJ+4;
%      S00=SF(I)*SF(J);
%      SX0=GDSF(1, I)*SF(J);
%      S0X=SF(I)*GDSF(1, J);
%      SY0=GDSF(2, I)*SF(J);
%      S0Y=SF(I)*GDSF(2, J);
%      SXX=GDSF(1, I)*GDSF(1, J);
%      SXY=GDSF(1, I)*GDSF(2, J);
%      SYX=GDSF(2, I)*GDSF(1, J);
%      SYY=GDSF(2, I)*GDSF(2, J);
%
%      ELK(II2, JJ2)=ELK(II2, JJ2)+CNST*(A55*SXX+A44*SYY+A45*(SXY+SYX));
%      ELK(II2, JJ3)=ELK(II2, JJ3)+CNST*(A55*SX0+A45*SY0);
%      ELK(II3, JJ2)=ELK(II3, JJ2)+CNST*(A55*S0X+A45*S0Y);
%      ELK(II2, JJ4)=ELK(II2, JJ4)+CNST*(A45*SX0+A44*SY0);
%      ELK(II4, JJ2)=ELK(II4, JJ2)+CNST*(A45*S0X+A44*S0Y);
%      ELK(II3, JJ3)=ELK(II3, JJ3)+CNST*A55*S00;
%      ELK(II3, JJ4)=ELK(II3, JJ4)+CNST*A45*S00;
%      ELK(II4, JJ3)=ELK(II4, JJ3)+CNST*A45*S00;
%      ELK(II4, JJ4)=ELK(II4, JJ4)+CNST*A44*S00;
%      JJ=NDF*J+1;
%
end
%      II=NDF*I+1;
%
end
end
end
end
%      if IDYN>0 & NEIGN==0
%      %
%      %      The Newmark intergration scheme for hyperbolic equations
%      %
%      %      if NCOUNT==1 & INTIAL~=0
%      %      for I = 1:NN
%      %      ELF(I) = 0.0;
%      %      for J = 1:NN

```

```

%             ELF(I) = ELF(I)-ELK(I,J)*ELU(J);
%             ELK(I,J)= ELM(I,J);
%         end
%     end
% else
%     for I = 1:NN
%         SUM = 0.0;
%         for J = 1:NN
%             SUM =
SUM+ELM(I,J)*(A3*ELU(J)+A4*ELV(J)+A5*ELA(J));
%             ELK(I,J)= ELK(I,J)+A3*ELM(I,J);
%         end
%         ELF(I) = ELF(I)+SUM;
%     end
% end
% end

```

### ELSTIF.m

```

function []=ELSTIF(M)
global E1 E2 G12 G13 G23 ANU12 RHO ALFA1 ALFA2 THETA THKNS MATYP
%Material
global Q QBAR ALFA Z
%Elastic
ANU21=ANU12(M)*E2(M)/E1(M);
DENOM=(1.0-ANU12(M)*ANU21);

Q(1,1)=E1(M)/DENOM;
Q(1,2)=ANU12(M)*E2(M)/DENOM;
Q(2,1)=Q(1,2);
Q(2,2)=Q(1,1)*E2(M)/E1(M);
Q(3,3)=G12(M);
Q(4,4)=G23(M);
Q(5,5)=G13(M);

```

### feaplyc2.m

```

function [kk,ff]=feaplyc2(kk,ff,ISPV,ISSV,NDF,NEQ,NSPV,NSSV,VSPV,VSSV)
for i=1:NSPV
    id=(ISPV(i,1)-1)*NDF+ISPV(i,2);
%     for j=1:NEQ
%         kk(c,j)=0;
%     end
%
%     kk(c,c)=1;
%     ff(c)=VSPV(i);
    value=VSPV(i);
    for j=1:NEQ
        ff(j)=ff(j)-value*kk(j,id);
        kk(id,j)=0;
        kk(j,id)=0;
    end
    kk(id,id)=1;
    ff(id)=value;
end

```

### feaplycs.m

```

function [kk,mm]=feaplycs(kk,mm,ISPV,NSPV,NEQ)

```

```

for i=1:NSPV
    c=ISPV(i);
    for j=1:NEQ
        kk(c,j)=0;
        kk(j,c)=0;
        mm(c,j)=0;
        mm(j,c)=0;
    end
    mm(c,c)=1;
end

feasmb11.m
function [kk]=feasmb11(kk,index)
global ELM % Stiff1
k=ELM;
for i=1:length(index)
    ii=index(i);
    for j=1:length(index)
        jj=index(j);
        kk(ii,jj)=kk(ii,jj)+k(i,j);
    end
end

feasmb12.m
function [kk,ff]=feasmb12(kk,ff,index)
global ELF ELK ELM % Stiff1
k=ELK;
f=ELF;
for i=1:length(index)
    ii=index(i);
    ff(ii)=ff(ii)+f(i);
    for j=1:length(index)
        jj=index(j);
        kk(ii,jj)=kk(ii,jj)+k(i,j);
    end
end

feeldof.m
function [index]=feeldof(NOD,NPE,NDF,N)
NN = NPE*NDF;
k=0;
for i=1:NPE
    start = (NOD(N,i)-1)*NDF;
    for j=1:NDF
        k=k+1;
        index(k)=start+j;
    end
end

femesh.m
function femesh();
for I=1:NNM
    II=NDF*(I-1)+1;
    JJ=II+NDF-1;
    GLFN(II)=GLFN(JJ);
    GLFN(II+1)=GLFN(JJ+1);
end

```

```

GLFN(II+2)=GLFN(II+2);
fprintf(LOUT, '\n %d \t', I);
X(I)=GLXY(I, 1);
Y(I)=GLXY(I, 2);
for kk=1:5
    for J=II:JJ
%       fprintf(LOUT, '%e\t', GLFN(J));
        disp(kk)=GLFN(J);
    end
end
XN(I)=X(I)+disp(1);
YN(I)=Y(I)+disp(2);
ZN(I)=disp(3);
end
[XX, YY, ZZ]=meshgrid(XN, YN, ZN);
surf(XX, YY, ZZ)

```

#### MSH2DR.m

```

function [NOD, GLXY, NNM, NEM]=MSH2DR( IEL, NX, NY, NPE, DX, DY, X0, Y0)
% IMPLICIT REAL*8 (A-H, O-Z)
%     DIMENSION NOD (MAXNEM, 9), GLXY (MAXNNM, 2), DX (MAXNX), DY (MAXNY)
%     COMMON/IO/IN, ITT
%
IELTYP=1;%always quadratic element
NEX1 = NX+1;
NEY1 = NY+1;
NXX  = IEL*NX;
NYY  = IEL*NY;
NXX1 = NXX + 1;
NYY1 = NYY + 1;
NEM  = NX*NY;
if IELTYP==0 NEM=2*NX*NY; end
NNM=NXX1*NYY1;
if NPE==8 NNM = NXX1*NYY1 - NX*NY; end
%
%
% if IELTYP==0
% Generate the array [NOD] and global coordinates [GLXY] for
% TRIANGULAR ELEMENTS
%
NX2=2*NX;
NY2=2*NY;
NOD(1,1) = 1;
NOD(1,2) = IEL+1;
NOD(1,3) = IEL*NXX1+IEL+1;
if NPE>3
    NOD(1,4) = 2;
    NOD(1,5) = NXX1 + 3;
    NOD(1,6) = NXX1 + 2;
end
%
NOD(2,1) = 1;
NOD(2,2) = NOD(1,3);
NOD(2,3) = IEL*NXX1+1;
if NPE>3
    NOD(2,4) = NOD(1,6);
    NOD(2,5) = NOD(1,3) - 1;
    NOD(2,6) = NOD(2,4) - 1;

```

```

end
%
K=3;
for IY=1:NY
  L=IY*NX2;
  M=(IY-1)*NX2;
  if NX>1
    for N=K:2:L
      for I=1:NPE
        NOD(N,I) = NOD(N-2,I)+IEL;
        NOD(N+1,I) = NOD(N-1,I)+IEL;
      end
    end
  end
  if IY<NY
    for I=1:NPE
      NOD(L+1,I)=NOD(M+1,I)+IEL*NXX1;
      NOD(L+2,I)=NOD(M+2,I)+IEL*NXX1;
    end
  end
  K=L+3
end
%
else
  %
  %
  %
  %
  %
  K0 = 0;
  if NPE==9 K0=1; end
  NOD(1,1) = 1;
  NOD(1,2) = IEL+1;
  NOD(1,3) = NXX1+(IEL-1)*NEX1+IEL+1;
  if NPE==9 NOD(1,3)=4*NX+5; end
  NOD(1,4) = NOD(1,3) - IEL;
  if NPE>4
    NOD(1,5) = 2;
    NOD(1,6) = NXX1 + (NPE-6);
    NOD(1,7) = NOD(1,3) - 1;
    NOD(1,8) = NXX1+1;
    if NPE==9
      NOD(1,9)=NXX1+2;
    end
  end
end
%
if NY>1
  M = 1;
  for N = 2:NY
    L = (N-1)*NX + 1;
    for I = 1:NPE
      NOD(L,I) = NOD(M,I)+NXX1+(IEL-1)*NEX1+K0*NX;
    end
    M=L;
  end
end
end

```

%

```
if NX>1
  for NI=2:NX
    for I = 1:NPE
      K1 = IEL;
      if I==6 | I==8    K1=1+K0; end
      NOD(NI, I) = NOD(NI-1, I)+K1;
    end
    M = NI;
    for NJ = 2:NY
      L = (NJ-1)*NX+NI;
      for J = 1:NPE
        NOD(L, J) = NOD(M, J)+NXX1+(IEL-1)*NEX1+K0*NX;
      end
      M = L;
    end
  end
end
end          %END IF FOR SELECTION OF RECT OR TRI
```

%

```
          Generate the global coordinates of the nodes, [GLXY]
DX(NEX1)=0.0;
DY(NEY1)=0.0;
XC=X0;
YC=Y0;
if NPE==8
  for NI = 1:NEY1
    I = (NXX1+NEX1)*(NI-1)+1;
    J = 2*NI-1;
    GLXY(I, 1) = XC;
    GLXY(I, 2) = YC;
    for NJ = 1:NX
      DELX=0.5*DX(NJ);
      I=I+1;
      GLXY(I, 1) = GLXY(I-1, 1)+DELX;
      GLXY(I, 2) = YC;
      I=I+1;
      GLXY(I, 1) = GLXY(I-1, 1)+DELX;
      GLXY(I, 2) = YC;
    end
  if NI<NY
    I = I+1;
    YC= YC+0.5*DY(NI);
    GLXY(I, 1) = XC;
    GLXY(I, 2) = YC;
    for II = 1:NX
      I = I+1;
      GLXY(I, 1) = GLXY(I-1, 1)+DX(II);
      GLXY(I, 2) = YC;
    end
  end
  YC = YC+0.5*DY(NI);
end
```

%



```

else
%
    YC=Y0;
    for NI = 1:NEY1
    XC = X0;
    I = NXX1*IEL*(NI-1);
    for NJ = 1:NEX1
    I=I+1;
    GLXY(I,1) = XC;
    GLXY(I,2) = YC;
    if NJ<NEX1
        if IEL==2
            I=I+1;
            XC = XC + 0.5*DX(NJ);
            GLXY(I,1) = XC;
            GLXY(I,2) = YC;
        end
    end
    XC = XC + DX(NJ)/IEL;
    end
    XC = X0;
    if IEL==2
        YC = YC + 0.5*DY(NI);
        for NJ = 1:NEX1
        I=I+1;
        GLXY(I,1) = XC;
        GLXY(I,2) = YC;
        if NJ<NEX1
            I=I+1;
            XC = XC + 0.5*DX(NJ);
            GLXY(I,1) = XC;
            GLXY(I,2) = YC;
        end
    end
    XC = XC + 0.5*DX(NJ);
    end
    end
    YC = YC + DY(NI)/IEL;
    end %for loop
end

```

### **piezo\_resultant.m**

```

function [SP]=piezo_resultant(p_tk, p_theta, EC31, EC32, EC36, EC14,
EC24, EC15, EC25, AV, NPLY,ITYPE)
%-----
% Calculates the piezo resutants PN, PM
%-----
% global P_tk p_theta EC13 EC23 EC33 AV NPLY
%Piezo
global PM PN
PN=zeros(3,1); PM=zeros(3,1);
for K=1:NPLY
    EP=[0; 0; -AV/p_tk];
    %Form EC matrix
    EC=zeros(3,3); % EC=zeros(3,6);
    ECBAR=zeros(3,3); %ECBAR=zeros(3,6);
    EC(3,1)=EC31; EC(3,2)=EC32;% EC(3,6)=EC36;

```

```

%Find ECBAR (Transformed Pieoelectric coefficients)
CN=cos(p_theta);
SN=sin(p_theta);
SN2=SN*SN;
CN2=CN*CN;

ECBAR(3,1)=EC(3,1)*CN2+EC(3,2)*SN2;
ECBAR(3,2)=EC(3,1)*SN2+EC(3,2)*CN2;
ECBAR(3,3)=EC(3,3);
% ECBAR(3,6)=(EC(3,1)-EC(3,2))*SN*CN;

%   if ITYPE>0
%       ECBAR(1,4)=(EC15-EC24)*SN*CN;
%       ECBAR(2,4)=EC24*CN2+EC15*SN2;
%       ECBAR(1,5)=EC15*CN2+EC24*SN2;
%       ECBAR(2,5)=(EC15-EC24)*SN*CN;
%   end
%
[QBARP]=QBARPIEZO(2,p_theta); %2 is the materials number later have
to be changed
ECBAR=ECBAR*QBARP;
ECBAR=ECBAR';
%Resultants
%for CLPT
for I=1:3
    for J=1:3
        PN(I)=PN(I)+ECBAR(I,J)*EP(J)*p_tk;
        PM(I)=PM(I)+ECBAR(I,J)*EP(J)*0.5*p_tk^2;
    end
end
%   PM=2*PM; PN=PN*2;
%for FSDT
if ITYPE>0
    SP=ECBAR(4:5,:)*EP*p_tk;
else
    SP=zeros(2,1);
end
end

```

#### **PLTSTF.m**

```

function []=PLTSTF(LAYERS,NSHR,IDYN,NTHERM,NPRNT,HSCALE)
%
% _____
% A,B,D,E,F,G,H: Laminate stiffnesses (integrals of Q(I,J)
% multiplied with 1,Z,Z2,Z3,Z4,Z5,Z6, respectively)
% ALFA(I): Thermal coefficients of expansion in the global coordinates
% (x,y,z).
% HT..... Total thickness of the laminate
% _____

global LINP LOUT %Files
global E1 E2 G12 G13 G23 ANU12 RHO ALFA1 ALFA2 THETA THKNS MATYP
%Material
global A B D E F G H A44 A45 A55 D44 D45 D55
%Stiff2
global Q QBAR ALFA Z
%Elastic

```

```

global P0 P1 P2 P3 P4 P5 P6                                %Integ

IDYN=1;
RHO=[7000 7000];

IT=6;
HT=0.0;
for I=1:LAYERS HT=HT+THKNS(I)*HSCALE; end
LL = LAYERS+1;
Z(1) = -0.5*HT;
for I=2:LL Z(I) = Z(I-1) + THKNS(I-1)*HSCALE; end

% In computing the laminate stiffnesses it is assumed that the
% midplane of the laminate is the xy-plane

for I=1:3
    if N THERM>0
        TN(I)=0.0;
        TM(I)=0.0;
    end
    for J=1:3
        A(I,J)=0.0;
        B(I,J)=0.0;
        D(I,J)=0.0;
    end
end
if NSHR==1
    A44=0.0;
    A45=0.0;
    A55=0.0;
end
if IDYN>0
    P0=0.0;
    P1=0.0;
    P2=0.0;
end
if NSHR>1
    for I=1:3
        if N THERM>0
            TL(I)=0.0;
            TK(I)=0.0;
        end
    end
    for J=1:3
        E(I,J)=0.0;
        F(I,J)=0.0;
        G(I,J)=0.0;
        H(I,J)=0.0;
    end
end

D44=0.0;
D45=0.0;
D55=0.0;
F44=0.0;
F45=0.0;

```

```

        F55=0.0;
        if IDYN>0
            P3=0.0;
            P4=0.0;
            P5=0.0;
            P6=0.0;
        end
    end
for K=1:LAYERS
    ANGLE = pi*THETA(K)/180.0;
    QBARSTF(K,ANGLE,NTHERM);
    if NPRNT~=0
        if K<=2
            if K==1
                fprintf(LOUT, '\n\n\t\tTRANSFORMED STIFFNESSES FOR THE
FIRST LAYER:\n\n');
            else
                fprintf(LOUT, '\n\n\n\t\t TRANSFORMED STIFFNESSES FOR
THE SECOND LAYER:\n\n');
            end
            fprintf(LOUT, '\n Elastic stiffness coefficients, [Q] \n');
for I=1:3
                fprintf(LOUT, '\n');
                for J=1:3 fprintf(LOUT, '\t %e', Q(I,J)); end
end
            fprintf(LOUT, '\n\tLamina stiffnesses, QBAR(I,J): \n');
for I=1:3
                fprintf(LOUT, '\n');
                for J=1:3 fprintf(LOUT, '\t %e',QBAR(I,J)); end
end
            if NSHR>=1
                fprintf(LOUT, '\n\tLamina stiffnesses, QBAR44, QBAR45,
QBAR55: \n');
                for I=4:5
                    fprintf(LOUT, '\n');
                    for J=4:5 fprintf(LOUT, '\t %e',QBAR(I,J)); end
end
            end
            if NTHERM>0
                for I=1:3 fprintf(LOUT, '\t %e',ALFA(I)); end
            end
        end
    end
end
%
% .....
% COMPUTE THE LAMINATE STIFFNESSES
% .....
% Laminate stiffnesses for the classical and first-order theories
ZTB=Z(K+1)-Z(K);
ZT2=Z(K+1)*Z(K+1);
ZB2=Z(K)*Z(K);
ZT3=ZT2*Z(K+1);
ZB3=ZB2*Z(K);
for I=1:3
    for J=1:3
        if NTHERM>0
            TN(I)=TN(I)+QBAR(I,J)*ALFA(J)*(T0*ZTB+0.5*T1*(ZT2-
ZB2));
            TM(I)=TM(I)+QBAR(I,J)*ALFA(J)*(0.5*T0*(ZT2-
ZB2)+T1*(ZT3-ZB3)/3.0);

```

```

        end
        A(I,J)=A(I,J)+QBAR(I,J)*ZTB;
        B(I,J)=B(I,J)+QBAR(I,J)*(ZT2-ZB2)/2.0;
        D(I,J)=D(I,J)+QBAR(I,J)*(ZT3-ZB3)/3.0;
    end
end

if IDYN>0
    M=MATYP(K);
    P0=P0+RHO(M)*ZTB;
    P1=P1+RHO(M)*(ZT2-ZB2)/2.0;
    P2=P2+RHO(M)*(ZT3-ZB3)/3.0;
end
if NSHR>0
    A44=A44+QBAR(4,4)*ZTB;
    A45=A45+QBAR(4,5)*ZTB;
    A55=A55+QBAR(5,5)*ZTB;
S(1,1)=A44;
S(1,2)=A45;
S(2,1)=A45;
S(2,2)=A55;
end
%
%
% Laminate stiffnesses for the third-order theory
%
if NSHR>1
    ZT4=ZT2*ZT2;
    ZB4=ZB2*ZB2;
    ZT5=ZT3*ZT2;
    ZB5=ZB3*ZB2;
    ZB6=ZB3*ZB3;
    ZT6=ZT3*ZT3;
    ZT7=ZT5*ZT2;
    ZB7=ZB5*ZB2;
for I=1:3
    for J=1:3
        if N THERM>0
            TL(I)=TL(I)+QBAR(I,J)*ALFA(J)*(T0*(ZT3-
ZB3)/3.0+T1*(ZT4-ZB4)/4.0);
            TK(I)=TK(I)+QBAR(I,J)*ALFA(J)*(T0*(ZT4-
ZB4)/4.0+T1*(ZT5-ZB5)/5.0);
        end
        E(I,J)=E(I,J)+QBAR(I,J)*(ZT4-ZB4)/4.0;
        F(I,J)=F(I,J)+QBAR(I,J)*(ZT5-ZB5)/5.0;
        G(I,J)=G(I,J)+QBAR(I,J)*(ZT6-ZB6)/6.0;
        H(I,J)=H(I,J)+QBAR(I,J)*(ZT7-ZB7)/7.0;
    end
end
D44=D44+QBAR(4,4)*(ZT3-ZB3)/3.0;
D45=D45+QBAR(4,5)*(ZT3-ZB3)/3.0;
D55=D55+QBAR(5,5)*(ZT3-ZB3)/3.0;
F44=F44+QBAR(4,4)*(ZT5-ZB5)/5.0;
F45=F45+QBAR(4,5)*(ZT5-ZB5)/5.0;
F55=F55+QBAR(5,5)*(ZT5-ZB5)/5.0;
if IDYN>0
    P3=P3+RHO(M)*(ZT4-ZB4)/4.0;
    P4=P4+RHO(M)*(ZT5-ZB5)/5.0;
end

```

```

        P5=P5+RHO(M)*(ZT6-ZB6)/6.0;
        P6=P6+RHO(M)*(ZT7-ZB7)/7.0;
    end
end
end

if NPRNT~=0
    fprintf(LOUT, '\n Laminate stiffnesses, [A]:\n' );
    for I=1:3
        fprintf(LOUT, '\n');
        for J=1:3 fprintf(LOUT, '\t %e', A(I,J)); end
    end
    fprintf(LOUT, '\n Laminate stiffnesses, [B]:\n ' );
    for I=1:3
        fprintf(LOUT, '\n');
        for J=1:3 fprintf(LOUT, '\t %e', B(I,J)); end
    end
    fprintf(LOUT, '\n Laminate stiffnesses, [D]:\n');
    for I=1:3
        fprintf(LOUT, '\n');
        for J=1:3 fprintf(LOUT, '\t %e', D(I,J)); end
    end
    if IDYN>0
        fprintf(LOUT, '\n Laminate mass inertias
(A0,A1,A2):\n ');
        fprintf(LOUT, '\t %e\t%e\t%e \n',P0,P1,P2);
    end

    if NSHR>0
        fprintf(LOUT, '\n\tLaminate shear stiffnesses
(A44,A45,A55):\n');
        fprintf(LOUT, '\t %e\t%e\t%e \n',A44,A45,A55);
    end

end

if NTHERM>0
    fprintf(LOUT, '\nThermal force resultants, NTXX, NTYY,
NTXY: ');

    for I=1:3 fprintf(LOUT, '\t %e',TN(I)); end
        fprintf(LOUT, '\n Thermal moment resultants, MTXX, MTYY,
MTXY: ');

        for I=1:3 fprintf(LOUT, '\t %e',TM(I)); end            end

    if NSHR>1
        fprintf(LOUT, '\n Laminate stiffnesses, [E]:\n');
    for I=1:3
        fprintf(LOUT, '\n');
        for J=1:3 fprintf(LOUT, '\t %e',E(I,J)); end
    end
    fprintf(LOUT, '\n Laminate stiffnesses, [F]:\n');
    for I=1:3
        fprintf(LOUT, '\n');
        for J=1:3 fprintf(LOUT, '\t %e',F(I,J)); end
    end
end

```

```

end
fprintf(LOUT, '\nLaminate stiffnesses, [G]:\n');
for I=1:3
    fprintf(LOUT, '\n');
    for J=1:3 fprintf(LOUT, '\t %e',G(I,J)); end
end
fprintf(LOUT, '\n Laminate stiffnesses, _H]:\n ');
for I=1:3
    fprintf(LOUT, '\n');
    for J=1:3 fprintf(LOUT, '\t %e',H(I,J)); end
end
fprintf(LOUT, '\n Laminate shear stiffnesses (D44,D45,D55):
\n');
fprintf(LOUT, '\t %e\t %e\t %e \n', D44,D45,D55);
fprintf(LOUT, '\n Laminate shear stiffnesses
(F44,F45,F55):\n');
fprintf(LOUT, '\t %e\t %e\t %e \n',F44,F45,F55);
if IDYN>0
    fprintf(LOUT, 'Laminate mass inertias
(A3,A4,A5,A6):\n');
    fprintf(LOUT, '\t %e\t %e\t %e\t %e\n', P3,P4,P5,P6);
end%if
if N THERM>0
    fprintf(LOUT, 'Higher order thermal resultants, L TXX,
L TYY, L TXY:\t');
    for I=1:3 fprintf(LOUT, '\t %e',TL(I)); end
    fprintf(LOUT, '\n Higher order thermal resultants, K TXX,
K TYY, K TXY:\t');
    for I=1:3 fprintf(LOUT, '\t %e',TK(I)); end
end
end
end
end

```

#### **QBARPIEZO.m**

```

function [QBARP]=QBARPIEZO(M,p_theta)
global E1 E2 G12 G13 G23 ANU12 RHO ALFA1 ALFA2 THETA THKNS MATYP
%Material

```

```

ANU21=ANU12(M)*E2(M)/E1(M);
DENOM=(1.0-ANU12(M)*ANU21);
QP(1,1)=E1(M)/DENOM;
QP(1,2)=ANU12(M)*E2(M)/DENOM;
QP(2,1)=QP(1,2);
QP(2,2)=QP(1,1)*E2(M)/E1(M);
QP(3,3)=G12(M);

```

```

ANGLE = pi*p_theta/180.0;

```

```

CN=cos(ANGLE);
SN=sin(ANGLE);
SN2=SN*SN;
SN3=SN2*SN;
SN4=SN2*SN2;
CN2=CN*CN;
CN3=CN2*CN;
CN4=CN2*CN2;

```

```

%
QBARP(1,1)=QP(1,1)*CN4+2.0*(QP(1,2)+2.0*QP(3,3))*SN2*CN2+QP(2,2)*SN4;
    QBARP(1,2)=(QP(1,1)+QP(2,2)-
4.0*QP(3,3))*SN2*CN2+QP(1,2)*(SN4+CN4);
    QBARP(1,3)=(QP(1,1)-QP(1,2)-2.0*QP(3,3))*SN*CN3+(QP(1,2)-
QP(2,2)+2.0*QP(3,3))*SN3*CN;
    QBARP(2,1)=(QP(1,1)+QP(2,2)-
4.0*QP(3,3))*SN2*CN2+QP(1,2)*(SN4+CN4);

QBARP(2,2)=QP(1,1)*SN4+2.0*(QP(1,2)+2.0*QP(3,3))*SN2*CN2+QP(2,2)*CN4;
    QBARP(2,3)=(QP(1,1)-QP(1,2)-2.0*QP(3,3))*SN3*CN+(QP(1,2)-
QP(2,2)+2.0*QP(3,3))*SN*CN3;
    QBARP(3,1)=(QP(1,1)-QP(1,2)-2.0*QP(3,3))*SN*CN3+(QP(1,2)-
QP(2,2)+2.0*QP(3,3))*SN3*CN;
    QBARP(3,2)=(QP(1,1)-QP(1,2)-2.0*QP(3,3))*SN3*CN+(QP(1,2)-
QP(2,2)+2.0*QP(3,3))*SN*CN3;
    QBARP(3,3)=(QP(1,1)+QP(2,2)-2.0*QP(1,2)-
2.0*QP(3,3))*SN2*CN2+QP(3,3)*(SN4+CN4);

```

#### **QBARSTF.m**

```

function []=QBARSTF(K,ANGLE,NTHERM)
global E1 E2 G12 G13 G23 ANU12 RHO ALFA1 ALFA2 THETA THKNS MATYP
global Q QBAR ALFA Z
%Elastic

M=MATYP(K);
ELSTIF(M);
CN=cos(ANGLE);
SN=sin(ANGLE);
SN2=SN*SN;
SN3=SN2*SN;
SN4=SN2*SN2;
CN2=CN*CN;
CN3=CN2*CN;
CN4=CN2*CN2;

%
QBAR(1,1)=Q(1,1)*CN4+2.0*(Q(1,2)+2.0*Q(3,3))*SN2*CN2+Q(2,2)*SN4;
QBAR(1,2)=(Q(1,1)+Q(2,2)-4.0*Q(3,3))*SN2*CN2+Q(1,2)*(SN4+CN4);
QBAR(1,3)=(Q(1,1)-Q(1,2)-2.0*Q(3,3))*SN*CN3+(Q(1,2)-
Q(2,2)+2.0*Q(3,3))*SN3*CN;
QBAR(2,1)=(Q(1,1)+Q(2,2)-4.0*Q(3,3))*SN2*CN2+Q(1,2)*(SN4+CN4);
QBAR(2,2)=Q(1,1)*SN4+2.0*(Q(1,2)+2.0*Q(3,3))*SN2*CN2+Q(2,2)*CN4;
QBAR(2,3)=(Q(1,1)-Q(1,2)-2.0*Q(3,3))*SN3*CN+(Q(1,2)-
Q(2,2)+2.0*Q(3,3))*SN*CN3;
QBAR(3,1)=(Q(1,1)-Q(1,2)-2.0*Q(3,3))*SN*CN3+(Q(1,2)-
Q(2,2)+2.0*Q(3,3))*SN3*CN;
QBAR(3,2)=(Q(1,1)-Q(1,2)-2.0*Q(3,3))*SN3*CN+(Q(1,2)-
Q(2,2)+2.0*Q(3,3))*SN*CN3;
QBAR(3,3)=(Q(1,1)+Q(2,2)-2.0*Q(1,2)-
2.0*Q(3,3))*SN2*CN2+Q(3,3)*(SN4+CN4);
QBAR(4,4)=Q(4,4)*CN2+Q(5,5)*SN2;
QBAR(4,5)=(Q(5,5)-Q(4,4))*CN*SN;
QBAR(5,4)=(Q(5,5)-Q(4,4))*CN*SN;
QBAR(5,5)=Q(5,5)*CN2+Q(4,4)*SN2;

```



```

    if NTHERM>0
% Transformation of the thermal coefficients of expansion
    ALFA(1)=ALFA1(M)*CN2+ALFA2(M)*SN2;
    ALFA(2)=ALFA1(M)*SN2+ALFA2(M)*CN2;
    ALFA(3)=2.0*(ALFA1(M)-ALFA2(M))*SN*CN;
end

SHPRCT.m
function [DET]=SHPRCT(NPE,XI,ETA,ELXY,ITYPE)
global SF GDSF SFH GDSFH GDDSFH %Shape
global P0 P1 P2 P3 P4 P5 P6 %Integ

XNODE =[-1.0 1.0 1.0 -1.0 0.0 1.0 0.0 -1.0 0.0 ; -1.0 -1.0 1.0 1.0 -1.0
0.0 1.0 0.0 0.0];
XNODE=XNODE';
NP=[1 2 3 4 5 7 6 8 9];
%FNC = A*B;
if NPE==4
% LINEAR Lagrange interpolation functions for FOUR-NODE element
for I = 1:NPE
    XP = XNODE(I,1);
    YP = XNODE(I,2);
    XI0 = 1.0+XI*XP;
    ETA0=1.0+ETA*YP;
    SF(I) = 0.25*XI0*ETA0;
    DSF(1,I)= 0.25*XP*ETA0;
    DSF(2,I)= 0.25*YP*XI0;
end
else
if NPE==8
% QUADRATIC Lagrange interpol. funct. for EIGHT-NODE element
(NPE=8)
%
for I = 1:NPE
    NI = NP(I);
    XP = XNODE(NI,1);
    YP = XNODE(NI,2);
    XI0 = 1.0+XI*XP;
    ETA0 = 1.0+ETA*YP;
    XI1 = 1.0-XI*XI;
    ETA1 = 1.0-ETA*ETA;
    if I<=4
        SF(NI) = 0.25*XI0*ETA0*(XI*XP+ETA*YP-1.0);
        DSF(1,NI) = 0.25*ETA0*XP*(2.0*XI*XP+ETA*YP);
        DSF(2,NI) = 0.25*XI0*YP*(2.0*ETA*YP+XI*XP);
    else
        if I<=6
            SF(NI) = 0.5*XI1*ETA0;
            DSF(1,NI) = -XI*ETA0;
            DSF(2,NI) = 0.5*YP*XI1;
        else
            SF(NI) = 0.5*ETA1*XI0;
            DSF(1,NI) = 0.5*XP*ETA1;
            DSF(2,NI) = -ETA*XI0;
        end
    end
end
end
end

```

```

end
end
end
else
%
% QUADRATIC Lagrange interpol. funct. for NINE-NODE element (NPE=9)
%
for I=1:NPE
NI = NP(I);
XP = XNODE(NI,1);
YP = XNODE(NI,2);
XI0 = 1.0+XI*XP;
ETA0 = 1.0+ETA*YP;
XI1 = 1.0-XI*XI;
ETA1 = 1.0-ETA*ETA;
XI2 = XP*XI;
ETA2 = YP*ETA;
if I<=4
SF(NI) = 0.25*XI0*ETA0*XI2*ETA2;
DSF(1,NI) = 0.25*XP*ETA2*ETA0*(1.0+2.0*XI2);
DSF(2,NI) = 0.25*YP*XI2*XI0*(1.0+2.0*ETA2);
else
if I <= 6
SF(NI) = 0.5*XI1*ETA0*ETA2;
DSF(1,NI) = -XI*ETA2*ETA0;
DSF(2,NI) = 0.5*XI1*YP*(1.0+2.0*ETA2);
else
if I <= 8
SF(NI) = 0.5*ETA1*XI0*XI2;
DSF(2,NI) = -ETA*XI2*XI0;
DSF(1,NI) = 0.5*ETA1*XP*(1.0+2.0*XI2);
else
SF(NI) = XI1*ETA1;
DSF(1,NI) = -2.0*XI*ETA1;
DSF(2,NI) = -2.0*ETA*XI1;
end
end
end
end %for
end
end
%
% Compute the Jacobian matrix [GJ] and its inverse [GJINV]
%
for I = 1:2
for J = 1:2
GJ(I,J) = 0.0;
for K = 1:NPE
GJ(I,J) = GJ(I,J) + DSF(I,K)*ELXY(K,J);
end
end
end
%
DET = GJ(1,1)*GJ(2,2)-GJ(1,2)*GJ(2,1);
GJINV(1,1) = GJ(2,2)/DET;
GJINV(2,2) = GJ(1,1)/DET;
GJINV(1,2) = -GJ(1,2)/DET;

```

```

GJINV(2,1) = -GJ(2,1)/DET;
%
% Compute the derivatives of the interpolation functions with
% respect to the global coordinates (x,y): [GDSF]
%
for I = 1:2
for J = 1:NPE
GDSF(I,J) = 0.0;
for K = 1:2
GDSF(I,J) = GDSF(I,J) + GJINV(I,K)*DSF(K,J);
end
end
end

if ITYPE<=1
if ITYPE==1
NET=4*NPE;
II = 1;
for I = 1:NPE
XP = XNODE(I,1);
YP = XNODE(I,2);
XI1 = XI*XP-1.0;
XI2 = XI1-1.0;
ETA1 = ETA*YP-1.0;
ETA2 = ETA1-1.0;
XI0 = (XI+XP)^2;
ETA0 = (ETA+YP)^2;
XIP0 = XI+XP;
XIP1 = 3.0*XI*XP+XP*XP;
XIP2 = 3.0*XI*XP+2.0*XP*XP;
YIP0 = ETA+YP;
YIP1 = 3.0*ETA*YP+YP*YP;
YIP2 = 3.0*ETA*YP+2.0*YP*YP;
%
SFH(II) = 0.0625*ETA0*ETA2*XI0*XI2;
DSFH(1,II) = 0.0625*ETA0*ETA2*XIP0*(XIP1-4.0);
DSFH(2,II) = 0.0625*XI0*XI2*YIP0*(YIP1-4.0);
DDSFH(1,II) = 0.125*ETA0*ETA2*(XIP2-2.0);
DDSFH(2,II) = 0.125*XI0*XI2*(YIP2-2.0);
DDSFH(3,II) = 0.0625*(XIP1-4.0)*(YIP1-4.0)*XIP0*YIP0;
%
SFH(II+1) = -0.0625*XP*XI0*XI1*ETA0*ETA2;
DSFH(1,II+1) = -0.0625*ETA0*ETA2*XP*XIP0*(XIP1-2.0);
DSFH(2,II+1) = -0.0625*XI0*XI1*XP*YIP0*(YIP1-4.);
DDSFH(1,II+1) = -0.125*ETA0*ETA2*XP*(XIP2-1.0);
DDSFH(2,II+1) = -0.125*XI0*XI1*(YIP2-2.0)*XP;
DDSFH(3,II+1) = -0.0625*XP*XIP0*(XIP1-2.)*(YIP1-4.)*YIP0;
%
SFH(II+2) = -0.0625*YP*XI0*XI2*(ETA0*ETA1);
DSFH(1,II+2) = -0.0625*ETA0*ETA1*YP*XIP0*(XIP1-4.);
DSFH(2,II+2) = -0.0625*XI0*XI2*YP*YIP0*(YIP1-2.);
DDSFH(1,II+2) = -0.125*ETA0*ETA1*YP*(XIP2-2.);
DDSFH(2,II+2) = -0.125*XI0*XI2*YP*(YIP2-1.0);
DDSFH(3,II+2) = -0.0625*YP*YIP0*(YIP1-2.)*(XIP1-4.0)*XIP0;
%
SFH(II+3) = 0.0625*XP*YP*XI0*XI1*(ETA0*ETA1);
DSFH(1,II+3) = 0.0625*ETA0*ETA1*XP*YP*(XIP1-2.)*XIP0;

```

```

DSFH(2, II+3) = 0.0625*XI0*XI1*XP*YP*(YIP1-2.)*YIP0;
DDSFH(1, II+3) = 0.125*ETA0*ETA1*XP*YP*(XIP2-1.);
DDSFH(2, II+3) = 0.125*XI0*XI1*XP*YP*(YIP2-1.0);
DDSFH(3, II+3) = 0.0625*XP*YP*YIP0*XIP0*(YIP1-2.)*(XIP1-2.);
II = I*4 + 1;
end
else
NET=3*NPE;
II = 1;
for I = 1:NPE
XP = XNODE(I,1);
YP = XNODE(I,2);
XI0 = XI*XP;
ETA0 = ETA*YP;
XIP1 = XI0+1;
ETAP1 = ETA0+1;
XIM1 = XI0-1;
ETAM1 = ETA0-1;
XID = 3.0+2.0*XI0+ETA0-3.0*XI*XI-ETA*ETA-2.0*XI/XP;
ETAD = 3.0+XI0+2.0*ETA0-XI*XI-3.0*ETA*ETA-2.0*ETA/YP;
ETAXI = 4.0+2.0*(XI0+ETA0)-3.0*(XI*XI+ETA*ETA)-
2.0*(ETA/YP+XI/XP);
%
SFH(II) = 0.125*XIP1*ETAP1*(2.0+XI0+ETA0-XI*XI-ETA*ETA);
DSFH(1, II) = 0.125*XP*ETAP1*XID;
DSFH(2, II) = 0.125*YP*XIP1*ETAD;
DDSFH(1, II) = 0.250*XP*ETAP1*(XP-3.0*XI-1.0/XP);
DDSFH(2, II) = 0.250*YP*XIP1*(YP-3.0*ETA-1.0/YP);
DDSFH(3, II) = 0.125*XP*YP*ETAXI;
%
SFH(II+1) = 0.125*XP*XIP1*XIP1*XIM1*ETAP1;
DSFH(1, II+1) = 0.125*XP*XP*ETAP1*(3.0*XI0-1.0)*XIP1;
DSFH(2, II+1) = 0.125*XP*YP*XIP1*XIP1*XIM1;
DDSFH(1, II+1) = 0.250*XP*XP*XP*ETAP1*(3.0*XI0+1.0);
DDSFH(2, II+1) = 0.0;
DDSFH(3, II+1) = 0.125*XP*XP*YP*(3.0*XI0-1.0)*XIP1;
%
SFH(II+2) = 0.125*YP*XIP1*ETAP1*ETAP1*ETAM1;
DSFH(1, II+2) = 0.125*XP*YP*ETAP1*ETAP1*ETAM1;
DSFH(2, II+2) = 0.125*YP*YP*XIP1*(3.0*ETA0-1.0)*ETAP1;
DDSFH(1, II+2) = 0.0;
DDSFH(2, II+2) = 0.250*YP*YP*YP*XIP1*(3.0*ETA0+1.0);
DDSFH(3, II+2) = 0.125*XP*YP*YP*(3.0*ETA0-1.0)*ETAP1;
II = I*3 + 1;
end
end

DDSF(1,1) = 0.0;
DDSF(2,1) = 0.0;
DDSF(3,1) = 0.250;
DDSF(1,2) = 0.0;
DDSF(2,2) = 0.0;
DDSF(3,2) = -0.250;
DDSF(1,3) = 0.0;
DDSF(2,3) = 0.0;
DDSF(3,3) = 0.250;
DDSF(1,4) = 0.0;

```

```

        DDSF(2,4) = 0.0;
        DDSF(3,4) = - 0.250;

        for I = 1:2
            for J = 1:NET
                SUM = 0.0;
                for K = 1:2
                    SUM = SUM + GJINV(I,K)*DSFH(K,J);
                end
                GDSFH(I,J) = SUM;
            end
        end
        for I = 1:3
            for J = 1:2
                SUM = 0.0D0;
                for K = 1:NPE
                    SUM = SUM + DDSF(I,K)*ELXY(K,J);
                end
                DJCB(I,J) = SUM;
            end
        end
        for K = 1:3
            for J = 1:NET
                SUM = 0.0;
                for L = 1:2
                    SUM = SUM + DJCB(K,L)*GDSFH(L,J);
                end
                DDSJ(K,J) = SUM;
            end
        end
        %
        % Compute the jacobian of the transformation
        %
        GGJ(1,1)=GJ(1,1)*GJ(1,1);
        GGJ(1,2)=GJ(1,2)*GJ(1,2);
        GGJ(1,3)=2.0*GJ(1,1)*GJ(1,2);
        GGJ(2,1)=GJ(2,1)*GJ(2,1);
        GGJ(2,2)=GJ(2,2)*GJ(2,2);
        GGJ(2,3)=2.0*GJ(2,1)*GJ(2,2);
        GGJ(3,1)=GJ(2,1)*GJ(1,1);
        GGJ(3,2)=GJ(2,2)*GJ(1,2);
        GGJ(3,3)=GJ(2,1)*GJ(1,2)+GJ(1,1)*GJ(2,2);

        GGINV=inv(GGJ); % CALL INVRSE(GGJ,GGINV);
        %
        for I = 1:3
            for J = 1:NET
                SUM = 0.0;
                for K = 1:3
                    SUM = SUM + GGINV(I,K)*(DSDFH(K,J)-DDSJ(K,J));
                end
                GDDSFH(I,J) = SUM;
            end
        end
    end
end

```

```

function
[ ]=STRESS (ELD, ELXY, HT, Q0, YL, ISTR, ITYPE, LAYERS, NDF, NPE, N THERM, I COUNT, MAX
ELM, MAXGPT, RNXX, RNY Y, RNXY, RMXX, RMY Y, RMX Y, RQX, RQY)
%
%
%   Laminate stresses are computed in this subroutine
%
%
RNXX=zeros (NEM, 9) ;
RNY Y=zeros (NEM, 9) ;
RNXY=zeros (NEM, 9) ;
RMXX=zeros (NEM, 9) ;
RMY Y=zeros (NEM, 9) ;
RMXY=zeros (NEM, 9) ;
RQX=zeros (NEM, 9) ;
RQY=zeros (NEM, 9) ;
ELD=ELU;
%
GAUSPT=[0.0 0.0 0.0 0.0 0.0;
        -0.57735027 0.57735027 0.0 0.0 0.0;
        -0.77459667 0.0 0.77459667 0.0 0.0;
        -0.86113631 -0.33998104 0.33998104 0.86113631 0.0;
        -0.90617984 -0.53846931 0.0 0.53846931 0.9061798];
GAUSPT=GAUSPT';

%

    %pi = 22/7;
    if ITYPE==0
        NET=3;
    else
        NET=4;
    end
    HT2=HT*HT;
    YL2=YL*YL;
    SCALE1=HT2/YL2/Q0;
    SCALE2=HT/YL/Q0;

%
IC=1;
for IGP=1:ISTR %700
%
for JGP=1:ISTR %600
%
    XI = GAUSPT (IGP, ISTR) ;
    ETA = GAUSPT (JGP, ISTR) ;
SHPRCT; % CALL SHPRCT (NPE, XI, ETA, DET, ELXY, ITYPE)
    XG=0.0;
    YG=0.0;
%C    DO 10 I=1, NPE
%C    XG=XG+ELXY (I, 1) *SF (I)
%C10   YG=YG+ELXY (I, 2) *SF (I)
    EXX0=0.0;
    EYY0=0.0;
    EXY0=0.0;

```

```

    EXX1=0.0;
    EYY1=0.0;
    EXY1=0.0;
    EXZ0=0.0;
    EYZ0=0.0;
%
%   Compute the membrane strains
for I=1:NPE
    L=(I-1)*NDF+1;
    EXX0=EXX0+GDSF(1,I)*ELD(L);
    EYY0=EYY0+GDSF(2,I)*ELD(L+1);
    EXY0=EXY0+(GDSF(2,I)*ELD(L)+(GDSF(1,I)*ELD(L+1)));
end
%
%   Compute bending strains
%
M=0;
for I=1:NPE
    J=(I-1)*NDF+1;
    if ITYPE<=1
        for K=1:NET
            L=J+K+1;
            M=M+1;
            EXX1=EXX1-GDDSFH(1,M)*ELD(L);
            EYY1=EYY1-GDDSFH(2,M)*ELD(L);
            EXY1=EXY1-2.0*GDDSFH(3,M)*ELD(L);
        end
    else
        EXX1=EXX1+GDSF(1,I)*ELD(J+3);
        EYY1=EYY1+GDSF(2,I)*ELD(J+4);
        EXY1=EXY1+GDSF(2,I)*ELD(J+3)+GDSF(1,I)*ELD(J+4);
        EXZ0=EXZ0+SF(I)*ELD(J+3)+GDSF(1,I)*ELD(J+2);
        EYZ0=EYZ0+SF(I)*ELD(J+4)+GDSF(2,I)*ELD(J+2);
    end %if
end
%
for L=1:LAYERS
    K=LAYERS-L+1;
    ANGLE = pi*THETA(K)/180.0;
QBARSTF; % CALL QBARSTF(K, ANGLE, N THERM)
    ZK1=Z(K);
    ZP1=Z(K+1);
%

SGMXXB=(QBAR(1,1)*(EXX0+EXX1*ZK1)+QBAR(1,2)*(EYY0+EYY1*ZK1)+QBAR(1,3)*(
EXY0+EXY1*ZK1))*SCALE1;

SGMXXT=(QBAR(1,1)*(EXX0+EXX1*ZP1)+QBAR(1,2)*(EYY0+EYY1*ZP1)+QBAR(1,3)*(
EXY0+EXY1*ZP1))*SCALE1;

SGMYYB=(QBAR(1,2)*(EXX0+EXX1*ZK1)+QBAR(2,2)*(EYY0+EYY1*ZK1)+QBAR(2,3)*(
EXY0+EXY1*ZK1))*SCALE1;

SGMYYT=(QBAR(1,2)*(EXX0+EXX1*ZP1)+QBAR(2,2)*(EYY0+EYY1*ZP1)+QBAR(2,3)*(
EXY0+EXY1*ZP1))*SCALE1;

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```

SGMXYB=(QBAR(1,3)*(EXX0+EXX1*ZK1)+QBAR(2,3)*(EYY0+EYY1*ZK1)+QBAR(3,3)*(
EXY0+EXY1*ZK1))*SCALE1;

SGMXYT=(QBAR(1,3)*(EXX0+EXX1*ZP1)+QBAR(2,3)*(EYY0+EYY1*ZP1)+QBAR(3,3)*(
EXY0+EXY1*ZP1))*SCALE1;
%      Transverse shear stresses for shear deformation theories
%      First-Order theory
      if ITYPE>1
          SGMYZT=(QBAR(4,4)*EYZ0+QBAR(4,5)*EXZ0)*SCALE2;
          SGMXZT=(QBAR(4,5)*EYZ0+QBAR(5,5)*EXZ0)*SCALE2;
      else
          SGMYZT=0;
          SGMXZT=0;
      end
%*****COMPUTATION OF RESULTANT FORCES AND MOMENTS FOR
LAMINATE*****
%
%      COMPUTATION OF Z
      ZP12=ZP1^2;
      ZK12=ZK1^2;
      ZP13=ZP1^3;
      ZK13=ZK1^3;
      ZINT1=ZP1-ZK1;
      ZINT2=0.5*(ZP12-ZK12);
      ZINT3=(ZP13-ZK13)/3.0;
%
%      COMPUTATION OF FORCE RESULTANTS
%
RNXX(ICOUNT,IC)=RNXX(ICOUNT,IC)+(QBAR(1,1)*(EXX0*ZINT1+EXX1*ZINT2)+QBAR
(1,2)*(EYY0*ZINT1+EYY1*ZINT2)+QBAR(1,3)*(EXY0*ZINT1+EXY1*ZINT2))*SCALE1
;
%
RNYY(ICOUNT,IC)=RNYY(ICOUNT,IC)+(QBAR(1,2)*(EXX0*ZINT1+EXX1*ZINT2)+QBAR
(2,2)*(EYY0*ZINT1+EYY1*ZINT2)+QBAR(2,3)*(EXY0*ZINT1+EXY1*ZINT2))*SCALE1
;
%
RNXY(ICOUNT,IC)=RNXY(ICOUNT,IC)+(QBAR(1,3)*(EXX0*ZINT1+EXX1*ZINT2)+QBAR
(2,3)*(EYY0*ZINT1+EYY1*ZINT2)+QBAR(3,3)*(EXY0*ZINT1+EXY1*ZINT2))*SCALE1
;
%
%      COMPUTATION OF MOMENT RESULTANTS
%
RMXX(ICOUNT,IC)=RMXX(ICOUNT,IC)+(QBAR(1,1)*(EXX0*ZINT2+EXX1*ZINT3)+QBAR
(1,2)*(EYY0*ZINT2+EYY1*ZINT3)+QBAR(1,3)*(EXY0*ZINT2+EXY1*ZINT3))*SCALE1
;
%
RMYY(ICOUNT,IC)=RMYY(ICOUNT,IC)+(QBAR(1,2)*(EXX0*ZINT2+EXX1*ZINT3)+QBAR
(2,2)*(EYY0*ZINT2+EYY1*ZINT3)+QBAR(2,3)*(EXY0*ZINT2+EXY1*ZINT3))*SCALE1
;
%

```





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7600
-254e-12 -254e-12 0 0 0 0 0
500e-3 500e-3
1
1 1
0.0 0.0 0.0
1 4 0 1 1
3 3 2
180 216
1 2 8 7
2 3 9 8
3 4 10 9
4 5 11 10
5 6 12 11
7 8 14 13
8 9 15 14
9 10 16 15
10 11 17 16
11 12 18 17
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0.5 0.270000011  
0.5 0.180000007  
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0.539939106 0.273494214  
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0.571195781 0.096228823  
0.586824059 0.007596124  
0.51710099 0.453015357  
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0.57866466 0.283870697  
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0.745134234 0.294307977  
0.814078212 0.23645708  
0.883022249 0.178606197  
0.543301284 0.474999994  
0.621243536 0.430000007  
0.699185848 0.38499999

0.7771281 0.340000004  
0.855070412 0.294999987  
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## PART2 BEAM

1. Run the BEAM.m
2. all the required input information in in the same file (change the values for different problems)
3. Uncomment the required sections, by default the code finds the optimal location of one actuator on a cantilever and plot the static deformations

### BEAM.m

```
% -----%
% Beam with piezoelctric actuators
% bc=1 -Simply supported
% bc=2 -fixed-free
% -----%

clc
clear all
nel=10;           % number of elements
nnel=2;          % number of nodes per element
ndof=2;          % number of dofs per node
nnode=(nnel-1)*nel+1; % total number of nodes in system
sdof=nnode*ndof; % total system dofs
bc=2;            %bc=1 Simply supported beam % bc=2%Fixed-free beam

el=79e9;         % elastic modulus
tleng=1;         % length of the beam
b=0.05; h=0.01; % bearth and thickness of the beam
xi=(b*h^3)/12;   % moment of inertia of cross-section
leng=tleng/nel; % element length of equal size
area=b*h;        % cross-sectional area of the beam
rho=2500;        % mass density
ipt=1;           % option for mass matrix
q0=0;            % distributed load
% m0=0;          % moment
% pl=0;          % point load
vol=0;           %Voltage
b=0.05; hp=0.01; tp=2e-4; d31=-254e-12;
Ep=63e9;         % elastic modulus of piezo

% el=70e9;       % elastic modulus
% xi=1.25e-12;   % moment of inertia of cross-section
% tleng=0.2;     % length of a half of the beam
% leng=tleng/nel; % element length of equal size
% area=1.5e-5;   % cross-sectional area of the beam
% rho=1;         % mass density
% ipt=1;         % option for mass matrix
% q0=2;          % distributed load
% m0=0;          % moment
% pl=0;          % point load
% vol=-80;       %Voltage
% b=0.015; hp=1e-3; tp=0.2e-3; d31=-254e-12;
% Ep=63e9;       % elastic modulus of piezo

% el=100e9;      % elastic modulus
% xi=1/12;       % moment of inertia of cross-section
% tleng=1;       % length of a half of the beam
```



```

% b=0.02; h=0.02;           % bearth and thickness of the beam
% xi=(b*h^3)/12;           % moment of inertia of cross-section
% leng=tleng/nel;         % element length of equal size
% area=b*h;               % cross-sectional area of the beam
% rho=1000;                % mass density
% ipt=1;                   % option for mass matrix
% q0=0;                    % distributed load
% % m0=0;                  % moment
% % pl=0;                  % point load
% vol=-80;                 %Voltage
% b=0.02; hp=0.01; tp=0.001; d31=-254e-12;
% Ep=63e9;                 % elastic modulus of piezo

bcdof(1)=1;                % first dof (deflection at left end) is constrained
bcval(1)=0;                % whose described value is 0
if bc==1
bcdof(2)=sdof-1;           % deflection at right end is constrained (Simply
Supported)
elseif bc==2;
bcdof(2)=2;                % Rotation at left end is constrained (Fixed-free)
end
bcval(2)=0;                % whose described value is 0

i=1:nel; piezo(i)=0;
%Piezoelectric Moment due to the voltage applied
% S Yang and B Ngoi, AIAA, Vol 38, No 12, pp 2292-2298
% mp=-d31*b*Ep*(hp+tp)*vol;
mp=-d31*b*Ep*(hp+tp);

ff=zeros(sdof,1);          % initialization of system force vector
kk=zeros(sdof,sdof);      % initialization of system matrix
mm=zeros(sdof,sdof);      % initialization of system matrix
index=zeros(nnel*ndof,1); % initialization of index vector

%Forming FE matrices
for iel=1:nel              % loop for the total number of elements
index=feeldof1(iel,nnel,ndof); % extract system dofs associated with
element
[k,m]=febeam1(el,xi,leng,area,rho,ipt); % compute element stiffness
matrix
f=feload(leng,q0,'l'); % compute element stiffness matrix - for
distributed

% % loading
% if q0>0
%     f=feload(leng,q0,'l');
% else
%     f=feload(leng,0,'l');
% end
% if piezo(iel)==1
%     fp=feload(leng,mp,'p');
% else
%     fp=feload(leng,0,'p');
% end
% f=f+fp;                  %element force matrix

% assemble each element matrix into system

```

```

[kk, ff]=feasmb12(kk, ff, k, f, index);
mm=feasmb11(mm, m, index);
end

% apply the boundary conditions
[kks, ffs]=feaplyc2(kk, ff, bcdof, bcval); %static
[kkd, mmd]=feaplycs(kk, mm, bcdof); %eigen
[kkk, mmm, fff]=aplybcdyn(bcdof, kk, mm, ff); %dyn

%Eigen values
[V1, D1]=eig(kkk, mmm);
nfreq=sqrt(D1);

%First order system
C=0.2*mmm+0.005*kkk; % Alpha=0.2, Beta=0.005
% C=0;
inv_m=inv(mmm);
A_mat = [zeros(size(kkk)) eye(size(kkk))
         -inv_m*kkk -inv_m*C];
C_mat=[eye(size(kkk)), zeros(size(kkk))];
F_mat= [zeros(length(kkk),1)
        inv_m*fff'];
Dsys=eig(A_mat);

% Modal Analysis
[Omega, Phi, ModF]=femodal(mmm, kkk, fff');
%Model Coordinate
kkk_m=Phi'*kkk*Phi;
mmm_m=Phi'*mmm*Phi;
%Eigen values
[V2, D2]=eig(kkk_m, mmm_m);
nfreq_m=sqrt(D2);

%First order system
% C=0.2*mmm_m+0.0004*kkk_m; % Alpha=0.2, Beta=0.005
C=0;
inv_m=inv(mmm_m);
A_mat_m = [zeros(size(kkk_m)) eye(size(kkk_m))
           -inv_m*kkk_m -inv_m*C];
% C_mat_m=[eye(size(kkk_m)), zeros(size(kkk_m))];
fff=Phi'*fff'; %Modal Coordinate
F_mat_m= [zeros(size(kkk_m),1)
          inv_m*fff];
Dsys_m=eig(A_mat_m);

C_mat_m=[eye(size(Phi))*Phi'*Phi', zeros(size(Phi))];

% %Forming Bact
for ielp=1:nel
fp=zeros(sdof,1); % initialization of system force vector
ffp=zeros(sdof-2,1); % initialization of system force vector
index=feeldof1(ielp, nnel, ndof); % extract system dofs associated with
element
f=feload(leng, mp, 'p');
fp=feasmb1f(fp, f, index);
%applying bc
[kkp, ffp]=feaplyc2(kk, ff, bcdof, bcval); %static

```

```

[kkk,mmm,ffp]=aplybcdyn(bcdof, kk, mm, fp);%dyn
% for i=1:sdof-2
%     if bc==2
%         ffp(i)=fp(i+2);
%     end
%     if bc==1
%         if i<sdof-1
%             ffp(i)=fp(i+1);
%         else
%             ffp(i)=fp(sdof);
%         end
%     end
% end
ffp=Phi'*ffp'; %Modal Coordinate
% ffp=ffp'; %Modal Coordinate
B_vec = [zeros(length(ffp),1)
inv_m*ffp];
Bact(:,ielp)=[B_vec];
end

% ttt=0:1/1947:1;
% sss=ss(A_mat_m,Bact(:,2),C_mat_m,0);
% figure(1)
% ires=impulse(sss);
% plot(ires(:,16))
% plot(ires(:,2),'g')
% plot(ttt,ires(:,1))
% hold on
% plot(ttt,ires(:,2),'g')
% hold off
%

% % % Swaping the column numbers
% % j=1;
% % for i=iel:-1:1
% %     Bactn(:,j)=Bact(:,i);
% %     j=j+1;
% % end
% % Bact=Bactn;

% %Call the optimization function
% %optactuator(A_mat,Bact,C,optact0,tol,itmax,type,cov);
%
[sigma,ub,lb,numit,optact,f,K,k]=optactuator(Phi,A_mat_m,Bact,C_mat_m,1
,0.000001,100,1,0);
% %
[sigma,ub,lb,numit,optact,f,K,k]=optactuator(A_mat,Bact,C_mat,5,0.00000
1,100,1,0);
% optlocation=optact(k);

% % %Hand calculation
% Q=C_mat_m'*C_mat_m;
% % Q=C_mat'*C_mat;
% R=1;
% for j=1:nel
%     B_mat=Bact(:,j);

```

```

% %      [f,K]=lqr(A_mat_m,B_mat,Q,R);
%      [f,K]=lqr(A_mat,B_mat,Q,R);
% [Vs,es]=eig(K);
% sigmahand(j)=.5*max(diag(es));
% end;
% opthand=min(sigmahand);
% optacthand=find(sigmahand<opthand+.00001)

% % % Hand calculation for controlability measure
% D=0;
% for j=1:nel
%     sys=ss(A_mat,Bact(:,j),C_mat,D);
%     WC=gram(sys,'c');
%     wc_tr(j)=trace(WC);
%     wc_det(j)=det(WC);
% end;

ploc=[3];
for pl=1:length(ploc)
%
vol=[100];
for vloop=1:length(vol)
% Optimal configuration
ff=zeros(sdof,1); % initialization of system force vector
kk=zeros(sdof,sdof); % initialization of system matrix
mm=zeros(sdof,sdof); % initialization of system matrix
index=zeros(nnel*ndof,1); % initialization of index vector
% i=1:nel; piezo(i)=1;

% piezo(1)=1; piezo(4)=1; piezo(6)=1;
piezo(ploc(pl))=1;
for iel=1:nel % loop for the total number of elements
index=feeldof1(iel,nnel,ndof); % extract system dofs associated with
element
[k,m]=febeam1(el,xi,leng,area,rho,ipt); % compute element stiffness
matrix
% loading
if q0>0
    f=feload(leng,q0,'l');
else
    f=feload(leng,0,'l');
end
if piezo(iel)==1
%     fp=feload(leng,mp,'p');
    fp=feload(leng,mp*vol(vloop),'p');
else
    fp=feload(leng,0,'p');
end
f=f+fp;
kk=feasmb11(kk,k,index); % assemble each element matrix into system
matrix
ff(nnode)=0;
[kk,ff]=feasmb12(kk,ff,k,f,index);
mm=feasmb11(mm,m,index); % assemble each element matrix into system
end
[kks,ffs]=feaplyc2(kk,ff,bcdof,bcval); % apply the boundary conditions
[kkd,mmd]=feaplycs(kk,mm,bcdof); % apply the boundary conditions

```

```

%Static analysis
fsol=kks\ffs; % Solve the matrix equation
% plotting the deformation
i=1:2:sdof-1;
j=1:nnode;
x=0:leng:tleng;
y(j)=fsol(i);
yv(:,vloop)=y(j);
end
hold on
figure(1)
plot(x,yv)

hold on
end

% % %Static analysis with one piezo at a time
for ielp=1:nel
i=1:nel; piezo(i)=0; piezo(ielp)=1;
ff=zeros(sdof,1); % initialization of system force vector
index=feeldof1(ielp,nnel,ndof); % extract system dofs associated with
element
load=mp; type='p';
f=feload(leng,load,type);
ff=feasmbly(ff,f,index);
% [kks,ffs]=feaplyc2(kk,ff,bcdof,bcval); % apply the boundary
conditions
fsols(:,ielp)=kks\ff; % Solve the matrix equation

if(ielp>1)
% plotting the deformation
i=1:2:sdof-1;
j=1:nnode;
x=0:leng:tleng;
y(j)=fsols(i,ielp);
figure(2)
hold on
plot(x,y)
end
end

% Dynamic analysis
%
% % % NOTE TO SELF
% A_mat_m = A_mat_m;
% B_mat_m = Bact(:,10);
% B_mat = Bact(:,6);
% C_mat_m = zeros(1,40);
% C_mat_m(20) = 1;
% D_mat_m = zeros(1,1);

t0 = 0;
t1 = 10;
nsteps1 = round((t1-t0)*200);
hsteps1 = (t1-t0)/nsteps1;
N=sdof-2;

```

```

% kk=kkk;
% mm=mmm;
% for i=1:sdof-2
%     fff(i)=ff(i+2);
% end
% ff=fff';
% % First order representation
% C=0.2*mm+0.005*kk; % Alpha=0.2, Beta=0.005
% % C=0;
% A_mat1 = [zeros(size(kk)) eye(size(kk))
%           -inv(mm)*kk      -inv(mm)*C];
% C_mat1=[eye(N), zeros(N)];
% B_mat1 = [zeros(size(ff))
%           inv(mm)*ff];
% D1=0;
% CO=ctrb(A_mat1,B_mat1);
% rank(CO)

B_mat=[Bact(:,6) F_mat_m];
% B_mat=[Bact(:,5) F_mat];
u=100;
d=1;
y0 = zeros(2*N,1);
tspan = t0:hsteps1:t1;
options =
odeset('JConstant','on','Jacobian','on','BDF','on','RelTol',1e-4,
'AbsTol',1e-4);
[t,y] = ode23s('yprime_beam',tspan,y0,options,A_mat_m,B_mat,u,d);
% [t,y] = ode23s('yprime_beam',tspan,y0,options,A_mat,B_mat,u,d);
yt1 = y(:,1:N);
figure(3)
% plot(t,yt1(:,end-1)) %uncontrolled ode23s
t=tspan;
u_new = [u*sin(t); d*sin(t)];
% sys = ss(A_mat, B_mat, C_mat,0);
sys = ss(A_mat_m, B_mat, C_mat_m,0);
% WC=gram(sys,'c');
[y,t] = lsim(sys,u_new,t);
% figure(4);
hold on;
plot(t,y(:,end-1),'r'); %uncontrolled lsim
% hold off;
xlabel('Time [s]');
ylabel('Amplitude [m]');

% optimal controll
% Q=C_mat'*C_mat;
Q=C_mat_m'*C_mat_m;
R=[1 0; 0 1];
% [f,K]=lqr(A_mat,B_mat,Q,R);
% A_new=(A_mat-Bact(:,6)*ones(1,length(B_mat))*K);
% syso = ss(A_new, F_mat, C_mat,0);
[f,K]=lqr(A_mat_m,B_mat,Q,R);
A_new=(A_mat_m-Bact(:,6)*ones(1,length(B_mat))*400000000*K); %
syso = ss(A_new, F_mat, C_mat_m,0);

```

```

[y,t] = lsim(syso,d*sin(t),t);
% figure(6);
% hold on;
plot(t,y(:,end-1),'g');

hold on;
% optimal control2
% Q=C_mat'*C_mat;
Q=C_mat_m'*C_mat_m;
R=[1 0; 0 1];
% [f,K]=lqr(A_mat,B_mat,Q,R);
% A_new=(A_mat-Bact(:,6)*ones(1,length(B_mat))*K);
% syso = ss(A_new, F_mat, C_mat,0);
% B_mat=[Bact(:,6) F_mat_m];
[f,K]=lqr(A_mat_m,B_mat,Q,R);
A_new=(A_mat_m-Bact(:,6)*ones(1,length(B_mat))*400000000*K);
syso = ss(A_new, F_mat, C_mat_m,0);
[y,t] = lsim(syso,2*sin(t),t);
% figure(6);
% hold on;
plot(t,y(:,end-1),'b');

hold off;

% Controllability tests
% CO=ctrb(A_mat_m,F_mat_m);
sys=ss(A_mat_m,B_mat,C_mat_m,0);
[sysb,g]=balreal(sys);
rsysb=balred(sysb,5);
CO=ctrb(rsysb.A,rsysb.B);
rank(CO);

% figure(4)
% plot(tspan,60*sin(tspan));
% figure(5)
% i=1;
% for ip=1:2:nel*2
%     yp(:,i)=y(:,ip);
%     i=i+1;
% end
% mesh(yp)
%
% figure(6)
% i=1;
% for ip=1:2:nel*2
%     ypt(:,i)=yt1(:,ip);
%     i=i+1;
% end
% mesh(ypt)

% x=1:10;
% y=t;
% z=yp(:,end-1);
% [xp,yp]=meshgrid(x,y);

```

```

% zp=griddata(x,y,z, xp, yp);
% surf(xp, yp, zp);
%
% % Q=C_mat_m'*C_mat_m;
% % R=1;
% % [f,K]=lqr(A_mat_m,B_mat_m,Q,R);
% % sysc = ss((A_mat_m-B_mat_m*K), B_mat_m, C_mat_m,0);
% % [y,t] = lsim(sys,60*sin(t),t);
% % figure(6)
% % plot(t,y(:,end-1),'g');

% % % Simulink
% tend = tspan(end);
% Ts = hsteps1;
% u = [tspan' 60*sin(tspan')];
% sim('beam_sim.mdl');
% plot(tspan, y(:,end-1));
% %
% %Step response
% figure(6)
% y=impulse(sys0);
% plot(y(:,2));
% figure(7)
% plot(y(:,3));
%plot(t,y(:,20))

```

### aplybcdyn.m

```

function [kkk,mmm,fff]=aplybcdyn(bcdof, kk, mm, ff)
%Apply BC for dynamic analysis - general way
kkk=kk;
mmm=mm;
fff=ff;
% Row
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpk = [kkk(bcdof(i)-i+2:end,:)];
        tmpm = [mmm(bcdof(i)-i+2:end,:)];
    elseif bcdof(i)-i+1 == size(kkk,1)
        tmpk = [kkk(1:bcdof(i)-i,:)];
        tmpm = [mmm(1:bcdof(i)-i,:)];
    else
        tmpk = [kkk(1:bcdof(i)-i,:); kkk(bcdof(i)-i+2:end,:)];
        tmpm = [mmm(1:bcdof(i)-i,:); mmm(bcdof(i)-i+2:end,:)];
    end
    kkk = tmpk;
    mmm = tmpm;
end
% Col
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpk = [kkk(:,bcdof(i)-i+2:end)];
        tmpm = [mmm(:,bcdof(i)-i+2:end)];
    elseif bcdof(i)-i+1 == size(kkk,2)
        tmpk = [kkk(:,1:bcdof(i)-i)];
        tmpm = [mmm(:,1:bcdof(i)-i)];
    else

```



```

        tmpk = [kkk(:,1:bcdof(i)-i) kkk(:,bcdof(i)-i+2:end)];
        tmpm = [mmm(:,1:bcdof(i)-i) mmm(:,bcdof(i)-i+2:end)];
    end
    kkk = tmpk;
    mmm = tmpm;
end
% F
for i = 1:length(bcdof)
    if bcdof(i)-i+1 == 1
        tmpf = [fff(bcdof(i)-i+2:end)];
    elseif bcdof(i)-i+1 == length(fff)
        tmpf = [fff(1:bcdof(i)-i)];
    else
        tmpf = [fff(1:bcdof(i)-i); fff(bcdof(i)-i+2:end)];
    end
    fff = tmpf;
end
fff=fff';

```

### feaplyc2.m

```

function [kk, ff]=feaplyc2(kk, ff, bcdof, bcval)
n=length(bcdof);
sdof=size(kk);

for i=1:n
    c=bcdof(i);
    for j=1:sdof
        kk(c, j)=0;
    end

    kk(c, c)=1;
    ff(c)=bcval(i);
end

```

### feaplycs.m

```

function [kk, mm]=feaplycs(kk, mm, bcdof)
n=length(bcdof);
sdof=size(kk);

for i=1:n
    c=bcdof(i);
    for j=1:sdof
        kk(c, j)=0;
        kk(j, c)=0;
        mm(c, j)=0;
        mm(j, c)=0;
    end

    mm(c, c)=1;
end

```

### feasmb11.m

```

function [kk]=feasmb11(kk, k, index)
edof = length(index);
for i=1:edof
    ii=index(i);

```

```

    for j=1:edof
        jj=index(j);
        kk(ii,jj)=kk(ii,jj)+k(i,j);
    end
end

```

### feasmb12

```

function [kk,ff]=feasmb12(kk,ff,k,f,index)
edof = length(index);
for i=1:edof
    ii=index(i);
    ff(ii)=ff(ii)+f(i);
    for j=1:edof
        jj=index(j);
        kk(ii,jj)=kk(ii,jj)+k(i,j);
    end
end

```

### feasmb1f.m

```

function [ff]=feasmb1f(ff,f,index)
edof = length(index);
for i=1:edof
    ii=index(i);
    ff(ii)=ff(ii)+f(i);
end

```

### feload.m

```

function [f]=feload(leng,load,type);
% This function forms the load vector
if type=='l'
    c=load/12;
    f=c*[6*leng; leng^2; 6*leng; -1*leng^2];
end
if type=='p'
    f=[0; load/2; 0; -load/2];
end
end

```

### febeam1.m

```

function [k,m]=febeam1(el,xi,leng,area,rho,ipt)
% stiffness matrix
c=el*xi/(leng^3);
k=c*[12      6*leng   -12      6*leng;...
     6*leng  4*leng^2 -6*leng  2*leng^2;...
     -12     -6*leng   12      -6*leng;...
     6*leng  2*leng^2 -6*leng  4*leng^2];

% consistent mass matrix

if ipt==1

    mm=rho*area*leng/420;
    m=mm*[156      22*leng   54      -13*leng;...
          22*leng  4*leng^2  13*leng  -3*leng^2;...

```

```

        54          13*leng    156          -22*leng;...
        -13*leng  -3*leng^2  -22*leng    4*leng^2];

% lumped mass matrix

elseif ipt==2

    m=zeros(4,4);
    mass=rho*area*leng;
    m=diag([mass/2  0  mass/2  0]);

% diagonal mass matrix

else

    m=zeros(4,4);
    mass=rho*area*leng;
    m=mass*diag([1/2  leng^2/78  1/2  leng^2/78]);

end

feeldof1.m
function [index]=feeldof1(iel,nnel,ndof)
edof = nnel*ndof;
start = (iel-1)*(nnel-1)*ndof;

for i=1:edof
    index(i)=start+i;
end

yprime_beam.m
function ydot = yprime_beam(t,y,flag,A_mat,B_mat,u,d)
switch flag
case ''
%     force = F_vec*1*sin(t);
    u_new = [u*sin(t'); d*sin(t')];
    force = B_mat*u_new;
%     force = B_mat(:,1)*u*sin(t) + B_mat(:,2)*d*sin(t);
%     force = F_vec*60;
    ydot = A_mat*y + force;

case 'jacobian'
    ydot = A_mat;
end

optactuator.m
function
[sigma,ub,lb,numit,optact,f,K,k]=optactuator(Phi,A,Bact,C,optact0,tol,i
tmax,type,cov);
% First pass at file to calculate optimal actuator locations
% 1 actuator. Columns of Bact are possible control operators.
% Pi0 is vector with entries 1 or 0 to indicate initial actuator
locations.
% Direct solution of lyapunov & ARE equations
% type=1, max eigenvalue; type =0 , trace

```

```

% % optactuator(A,Bact,C,1,0.001,100,1,0);
[N,M]=size(Bact);
R=eye(M);
% C=Phi'*C*Phi;
Q=C'*C;
%
% Step 1
% Initialization
%
k=1;
Pi=zeros(M,1);
Pi(optact0)=1;
optact(1)=optact0;
B=Bact(:,optact(1));

[f,K]=lqr(full(A),full(B),full(Q),1);
if(type==1)
[V,es]=eig(K);
zeta=V(:,N);
sigma(k)=.5*max(eig(K));
else
zeta=cov;
sigma(k)=.5*trace(K*cov);
end;
ub(k)=sigma(k);
lb(k)=0;
err=(ub(k)-lb(k))/ub(k);

relcon=zeros(M,1);

%
% Optimization loop
%
while(err>tol & k<itmax)
    % Step 2 solve master problem
    ZZ=lyap(A-B*f,zeta*zeta');
    S=-1/2*K*ZZ*K;
    for j=1:M
        grad(j)=trace(Bact(:,j)*(Bact(:,j)'+S));
    end;
    d=sigma(k)-grad*Pi;
    %
    % alg. for 1 actuator using (65) in Geromel
    % first find max over all previous calc., calc. at 1st location
    for j=1:M;
        relcon(j)=max(relcon(j),d(:)+grad(:,j));
    end;
    lb(k+1)=min(relcon);
    index=find(relcon<lb(k+1)+tol*.001);
    optact(k+1)=index(1);
    %
    % Now have current best location, and lower bound on cost
    %
    % Step 3
    % Update cost with current best location
    Pi=zeros(M,1);

```

```

Pi(optact(k+1))=1;
B=sparse(Bact(:,optact(k+1)));
% Using 0 as initial estimate
%[f,cP,Ricounter,Lycounter,TL]=sparsenk2(A,B,C,sparse(zeros(1,6)),nktol
*.01
[f,K]=lqr(full(A),full(B),full(Q),1);
if(type==1) % use lambda max

[V,es]=eig(K);
zeta=V(:,N);
sigma(k+1)=.5*max(eig(K));
else
    sigma(k+1)=.5*trace(K*cov);
end;
if(ub>sigma(k+1))
    ub=sigma(k+1);
end;
err=(sigma(k+1)-lb(k+1))/sigma(k+1);
k=k+1;
% Repeat if upper bound currentbd and lower bound theta not close.
end;
if(k==itmax)
    disp('WARNING: maximum number of iterations for actuator location
reached.')
end;
numit=k;
disp('Optimal location is')
disp(optact(k))
%
```